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**Miller**

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(54) **ROOM AIR PURIFIER WITH  
PRESSURIZATION RELIEF**

(56) **References Cited**

U.S. PATENT DOCUMENTS

(71) Applicant: **Gregory R. Miller**, Washington, MO  
(US)

3,618,659 A \* 11/1971 Rawal ..... 165/248  
4,336,748 A 6/1982 Martin et al.  
4,940,475 A \* 7/1990 Yaeger ..... 96/258  
5,099,836 A 3/1992 Rowland et al.

(Continued)

(72) Inventor: **Gregory R. Miller**, Washington, MO  
(US)

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FOREIGN PATENT DOCUMENTS

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OTHER PUBLICATIONS

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Safe Room with IQAir Purifiers with Duct Kits, Website  
AllergyBuyersClub.com, Oct. 22, 2008, 6 pgs.

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*Primary Examiner* — Robert A Hopkins

*Assistant Examiner* — Minh-Chau Pham

(74) *Attorney, Agent, or Firm* — Morgan, Lewis & Bockius  
LLP

(52) **U.S. Cl.**

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(2013.01); **B01D 46/0086** (2013.01); **B01D**  
**46/446** (2013.01); **B01D 46/448** (2013.01);  
**F24F 3/1603** (2013.01); **F24F 11/022**  
(2013.01); **F24F 2003/1682** (2013.01); **F24F**  
**2011/0042** (2013.01)

(57)

**ABSTRACT**

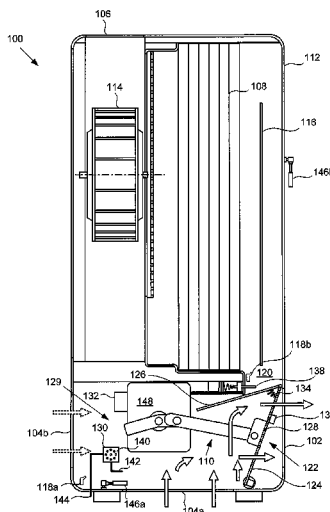
Systems and methods are provided for an air purifier with  
over pressurization relief, including the following compo-  
nents. A room air inlet is configured to fluidly receive air from  
a room. A supply duct inlet is configured to fluidly receive air  
from a forced air system. An air filter and or air purification  
means is configured to filter/purify air. An outlet is configured  
to expel filtered/treated air into the room. A bypass damper  
system has at least one air filtering mode position and a  
bypass mode position. A pressure control mechanism is con-  
figured to automatically allow the bypass damper system to  
move from at least one air filtering mode position to a bypass  
mode position when the air received from the supply duct  
inlet exceeds a bypass threshold. The bypass mode position  
guides air from the supply duct inlet into the room without  
requiring passage through the filter and the outlet.

(58) **Field of Classification Search**

CPC ..... B01D 29/62; B01D 35/143; B01D 35/147  
USPC ..... 55/312, 432, 465, 428, 429, 431, 467.1,  
55/466, DIG. 34; 95/273, 19; 96/414, 416,  
96/418, 400, 421

See application file for complete search history.

**39 Claims, 22 Drawing Sheets**



(56)

**References Cited**

U.S. PATENT DOCUMENTS

5,564,626	A *	10/1996	Kettler et al. ....	236/49.3
5,884,500	A	3/1999	Wetzel	
6,221,314	B1	4/2001	Bigelow	
6,500,387	B1	12/2002	Bigelow	
6,783,578	B2	8/2004	Tillman, Jr.	
6,960,241	B1 *	11/2005	Slenz .....	95/19
7,105,037	B2	9/2006	Olander et al.	
7,326,387	B2	2/2008	Arts et al.	
7,332,006	B2	2/2008	Kim et al.	
7,578,734	B2	8/2009	Ahmed et al.	
8,118,236	B2	2/2012	Lestage et al.	

8,568,521	B2 *	10/2013	Sakashita et al. ....	96/418
2004/0058637	A1	3/2004	Laiti	
2005/0164625	A1	7/2005	Kim et al.	
2006/0021375	A1	2/2006	Wetzel et al.	
2006/0177356	A1	8/2006	Miller	
2008/0076345	A1	3/2008	Wobben	
2009/0023376	A1	1/2009	Miller	
2010/0044447	A1	2/2010	Miller	

OTHER PUBLICATIONS

Airehaven, Inc., International Search Report and Written Opinion,  
PCT/US2014/055912, Nov. 25, 2014, 8 pgs.

\* cited by examiner

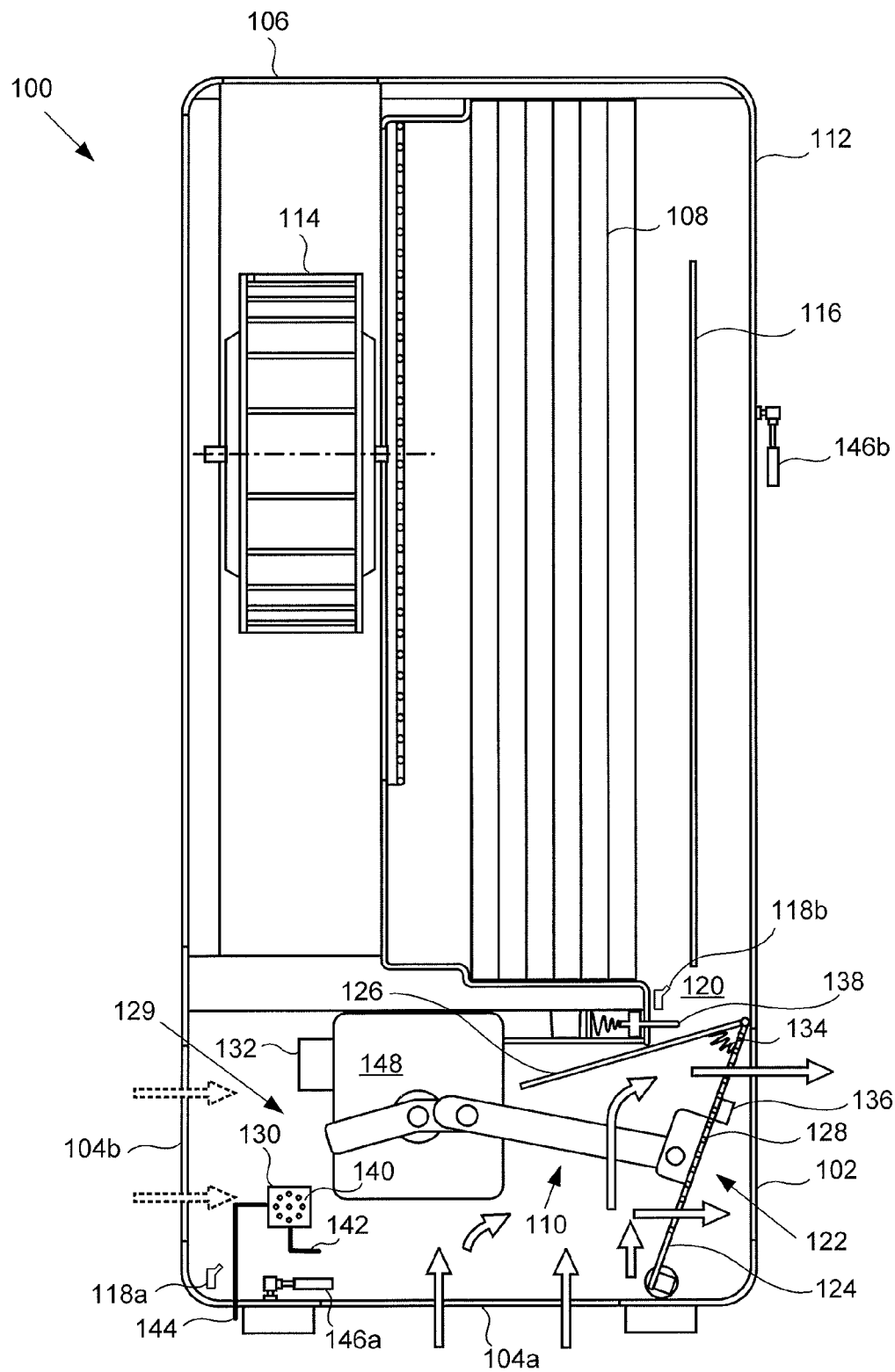


FIG. 1

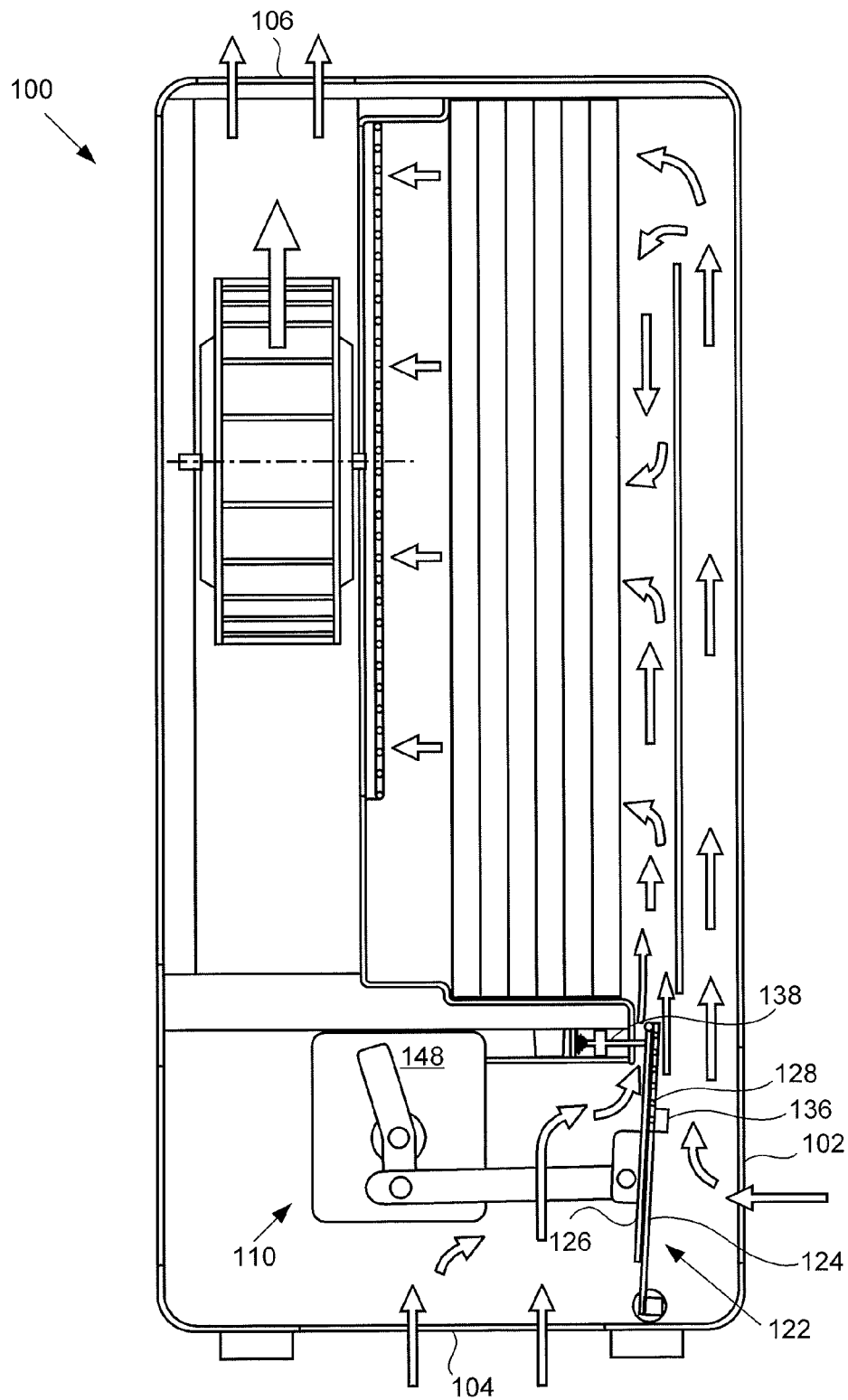


FIG. 2A

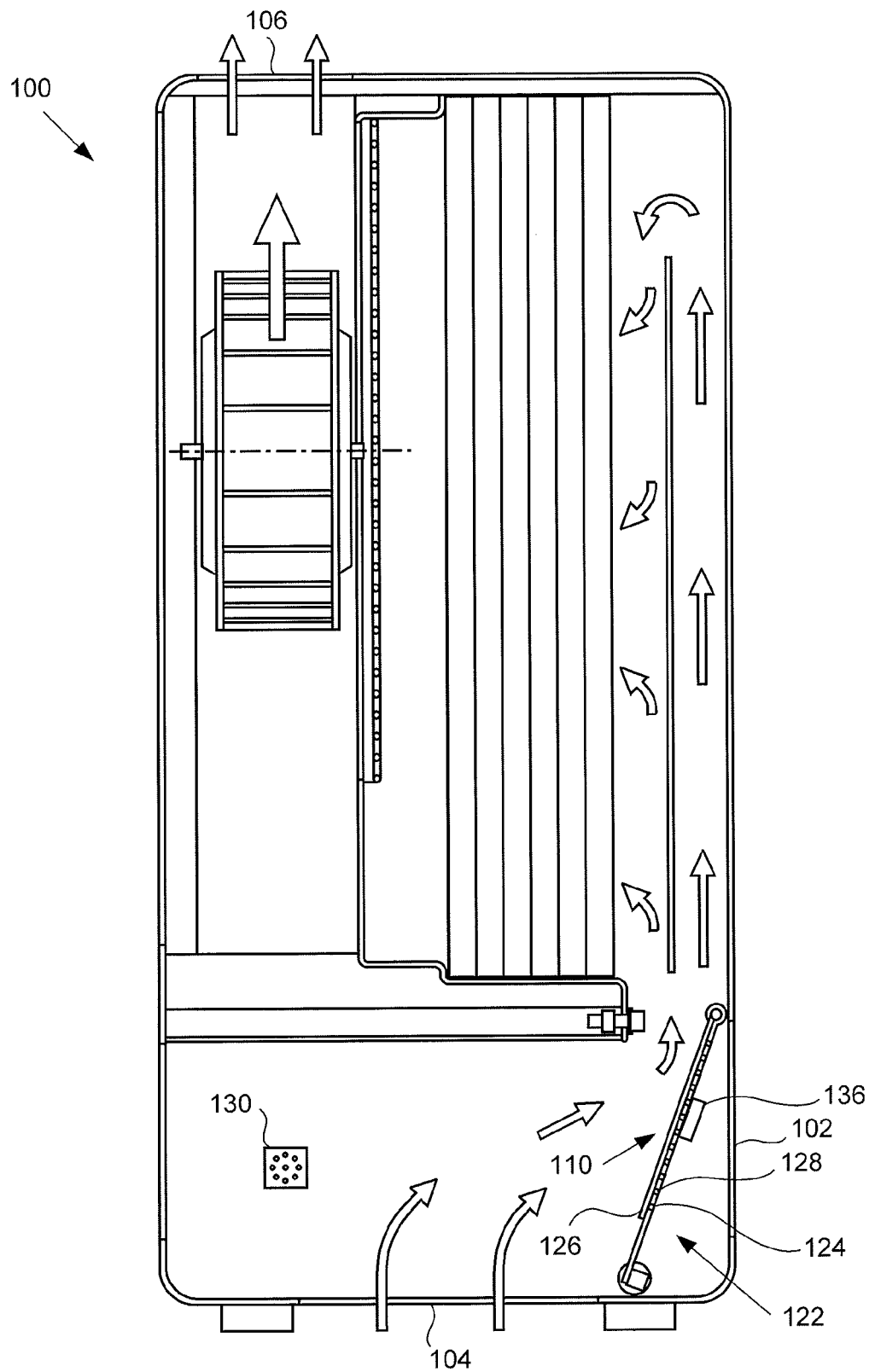


FIG. 2B

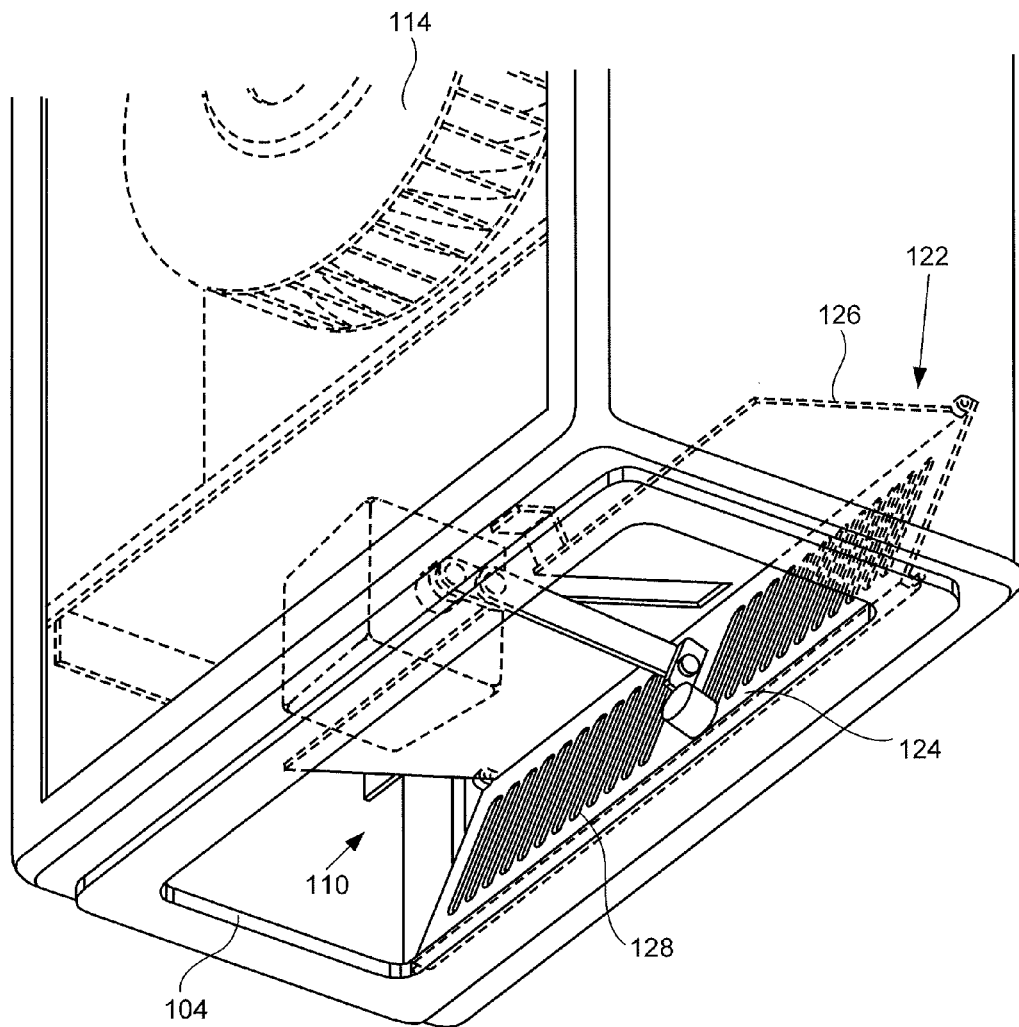


FIG. 2C

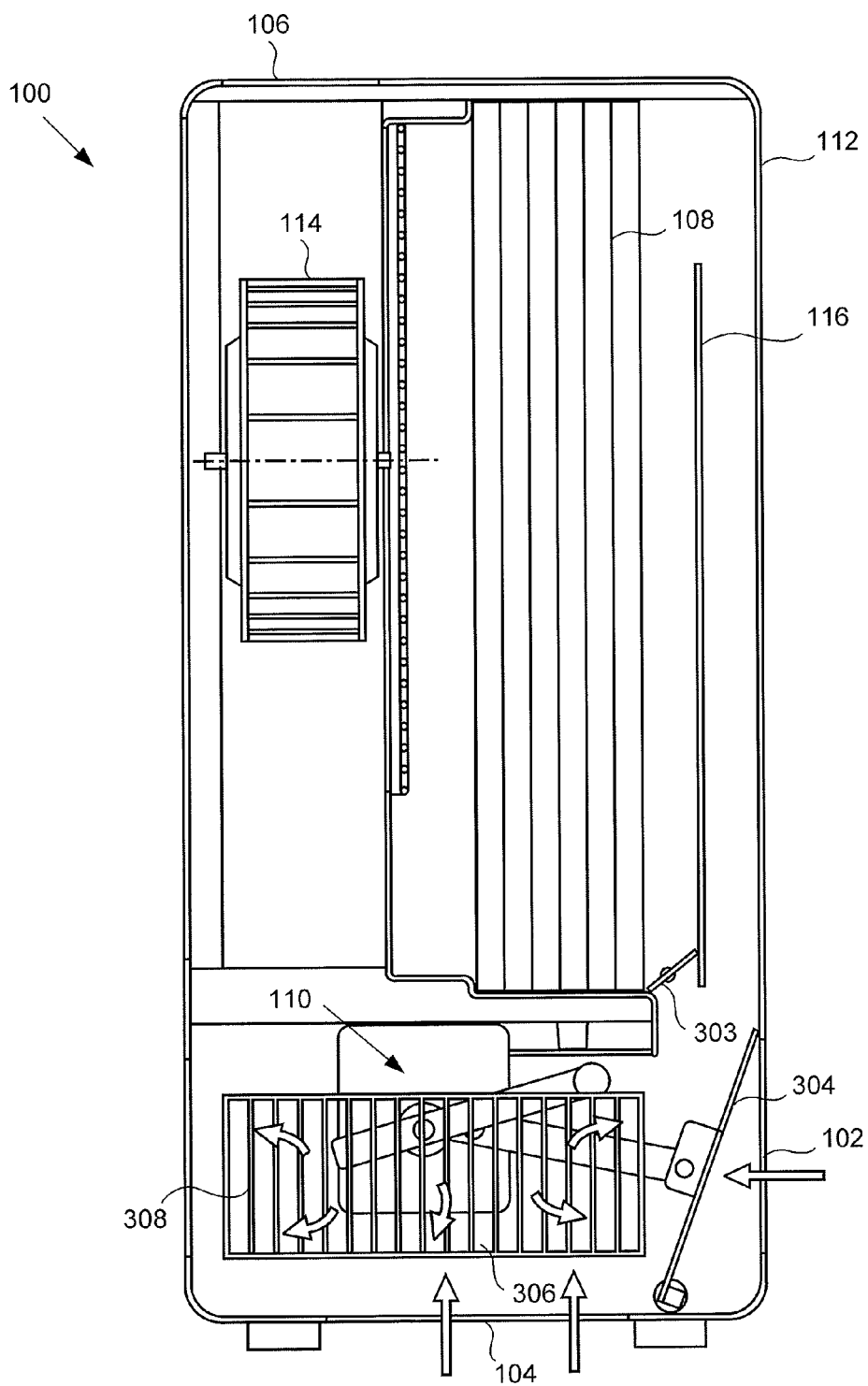


FIG. 3

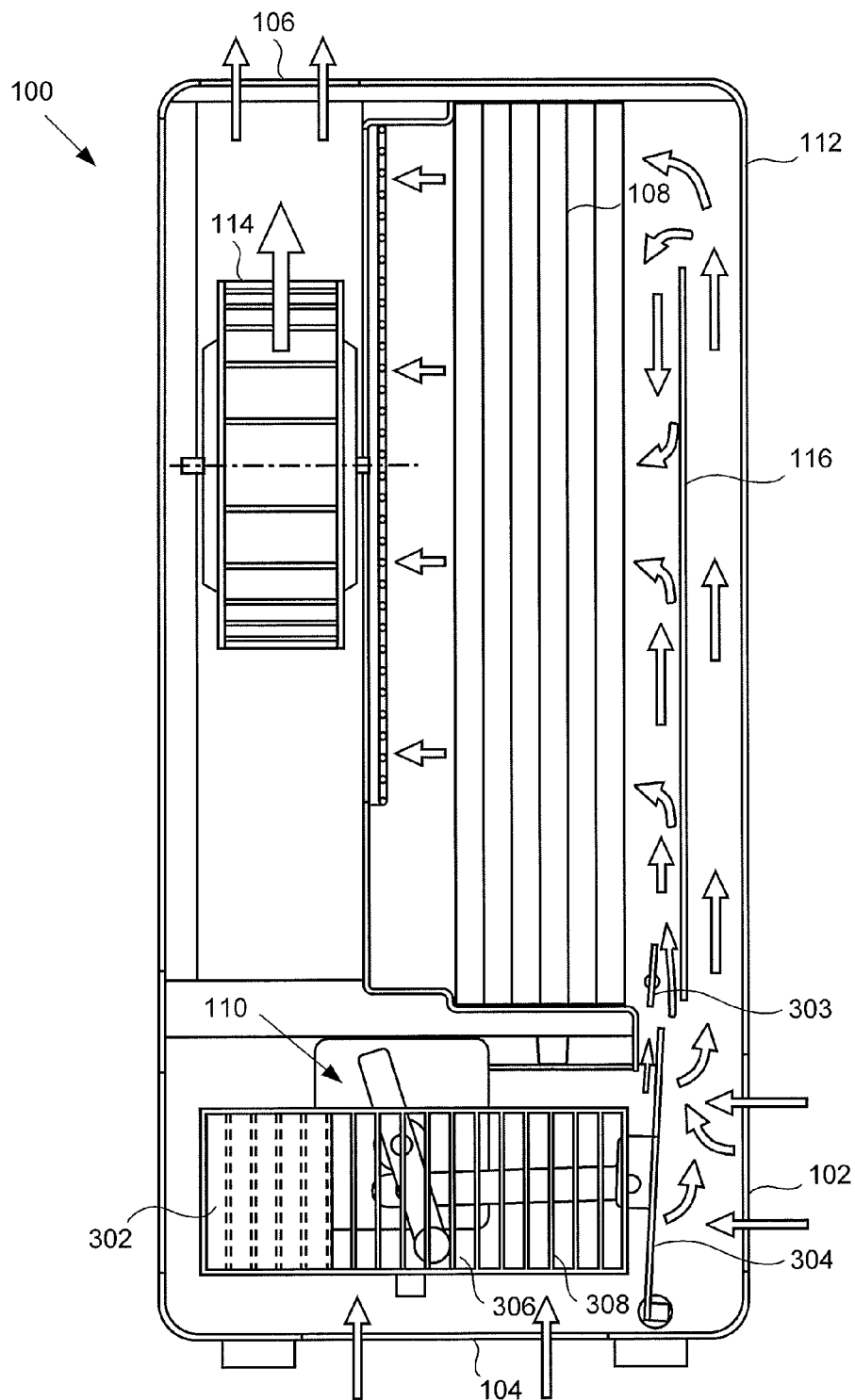


FIG. 4



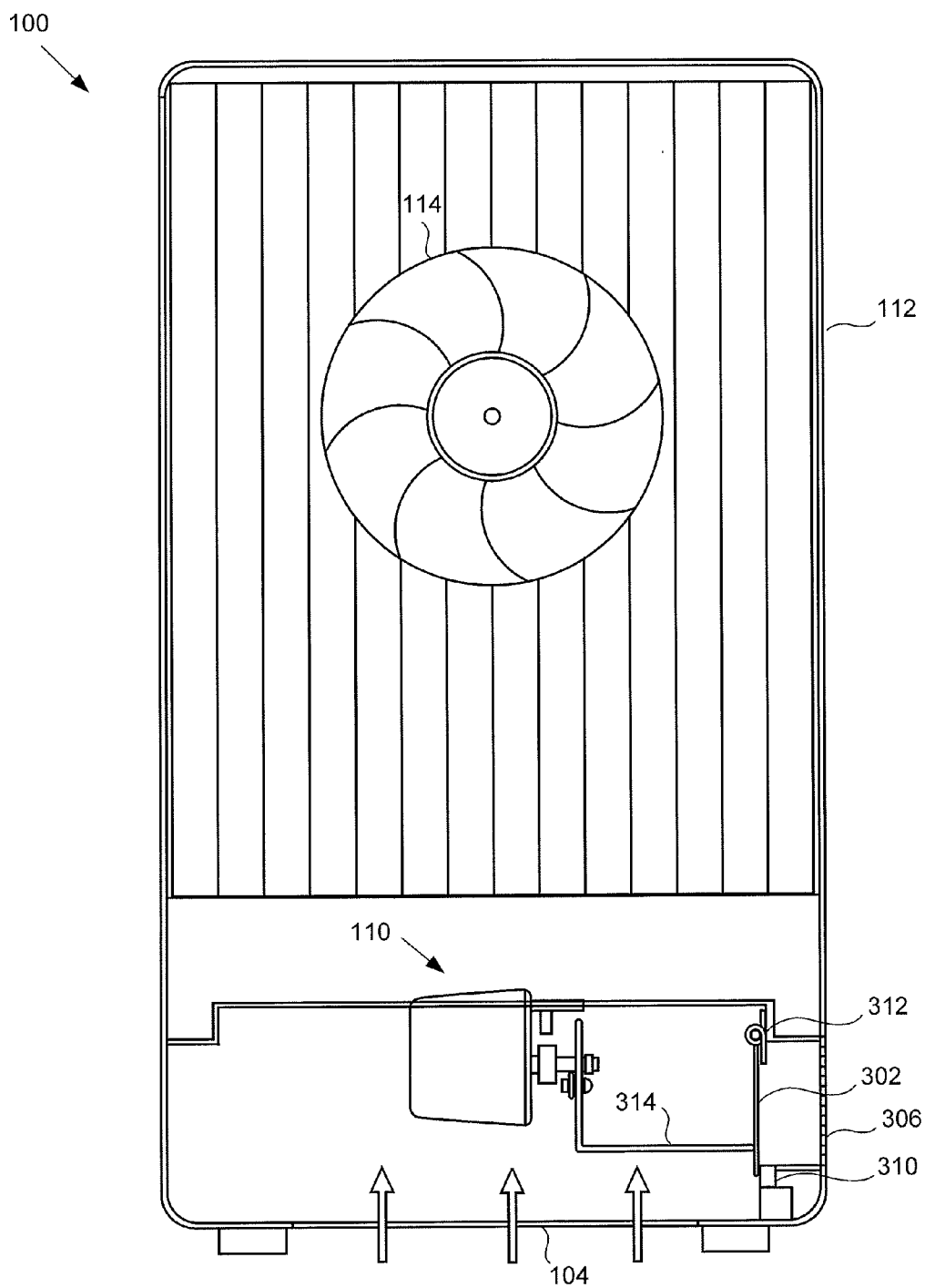


FIG. 5

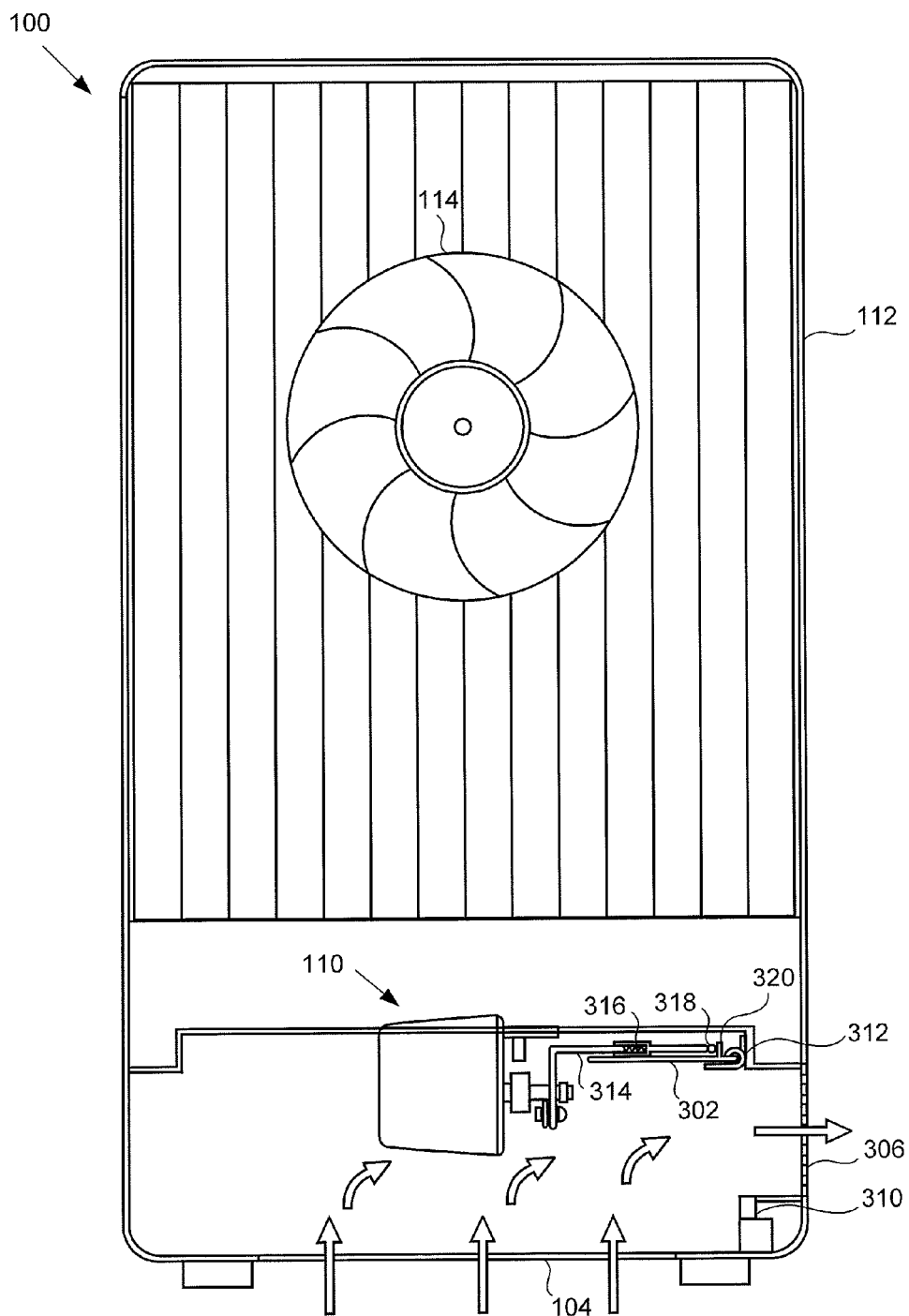


FIG. 6

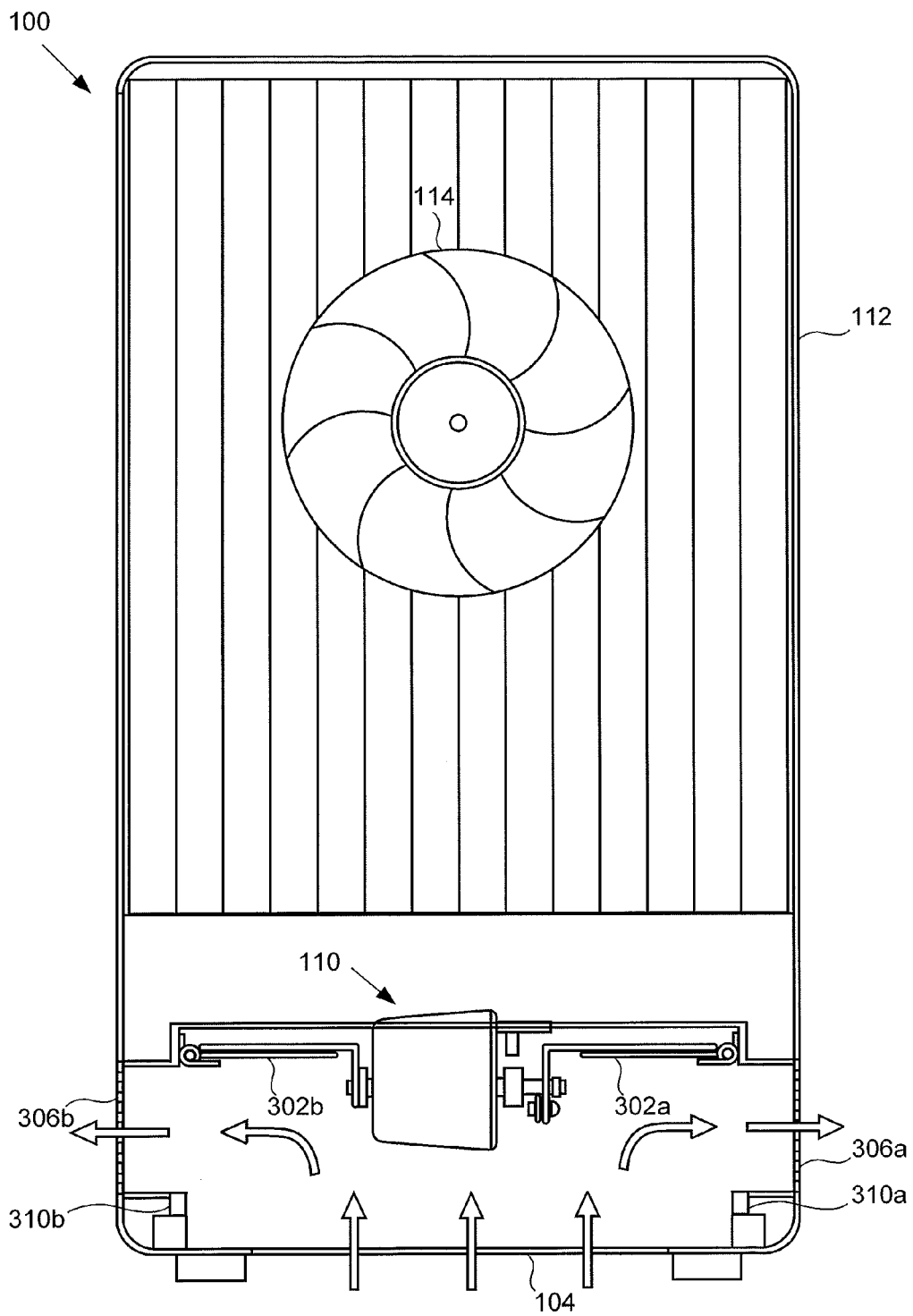


FIG. 7

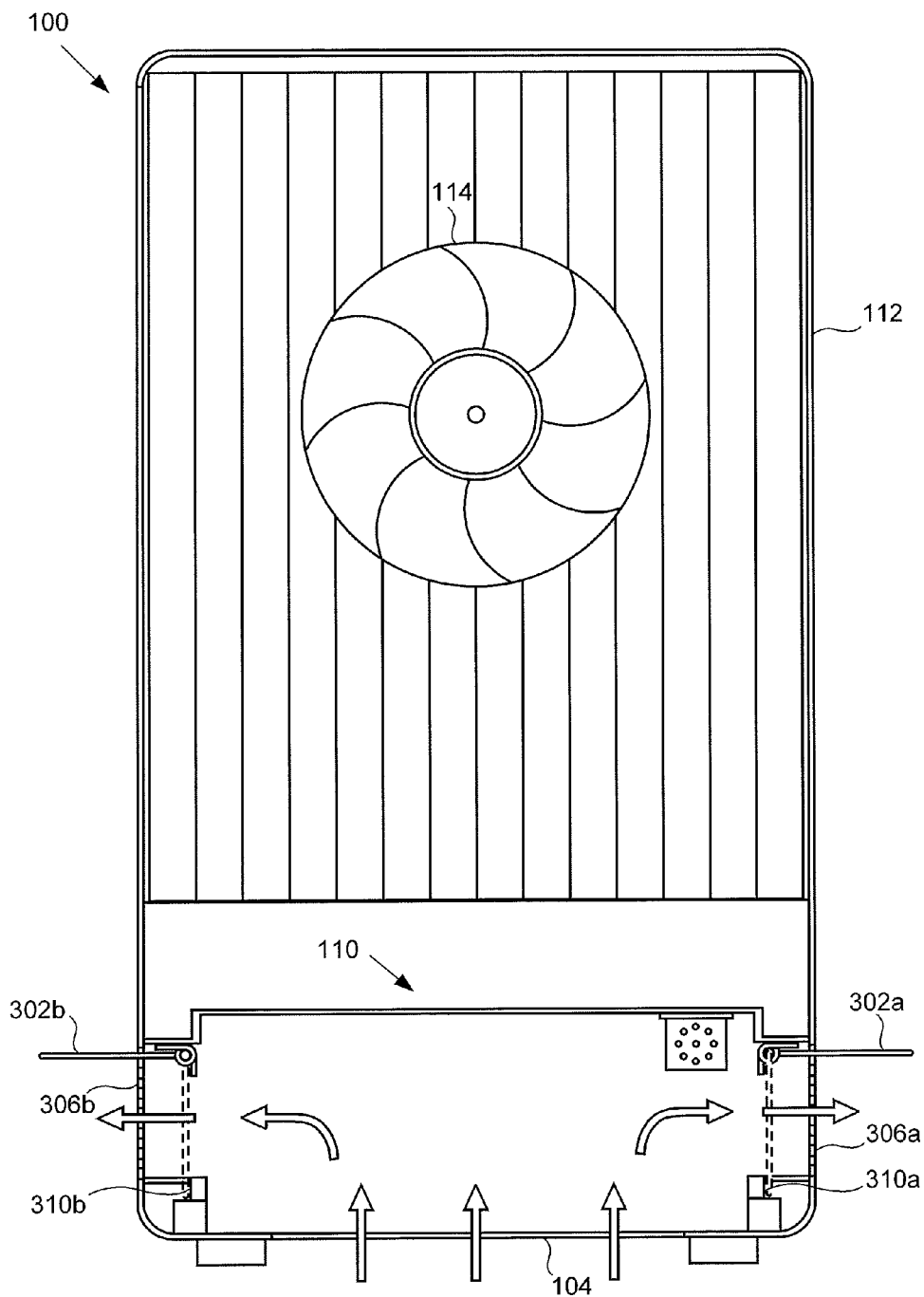


FIG. 8

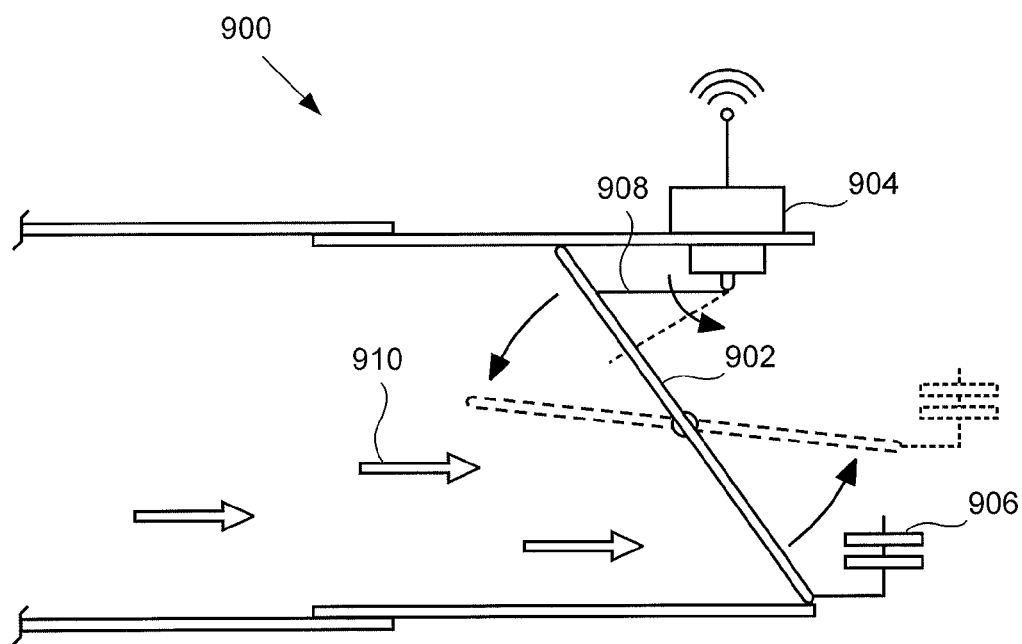


FIG. 9

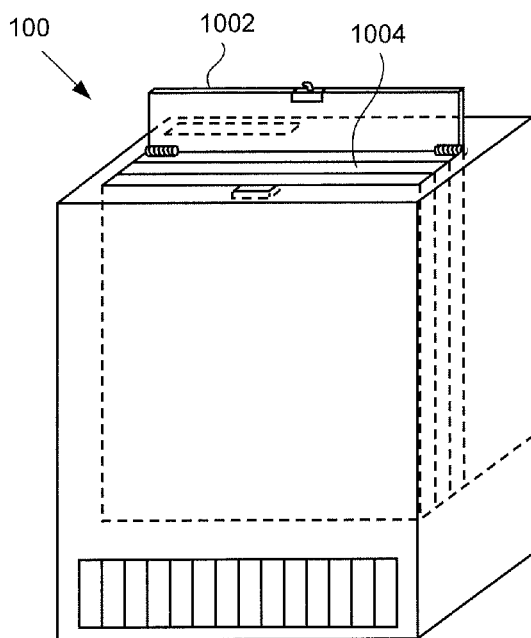


FIG. 10A

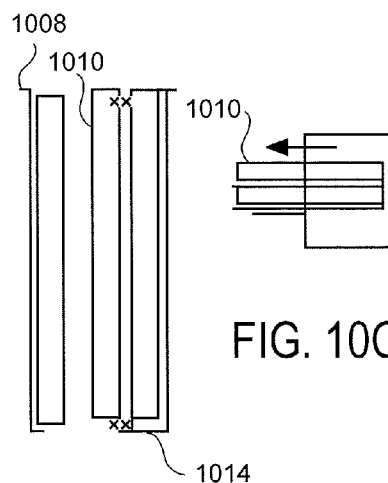


FIG. 10B

FIG. 10C

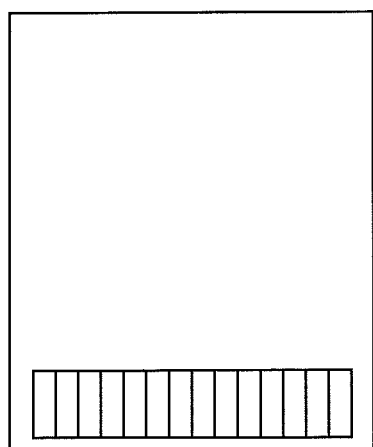


FIG. 10E

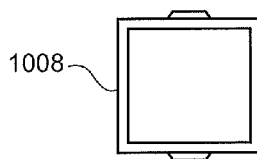


FIG. 10D

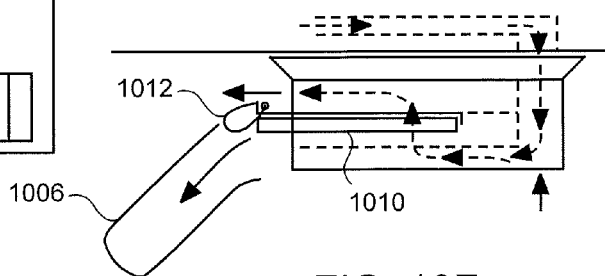


FIG. 10F

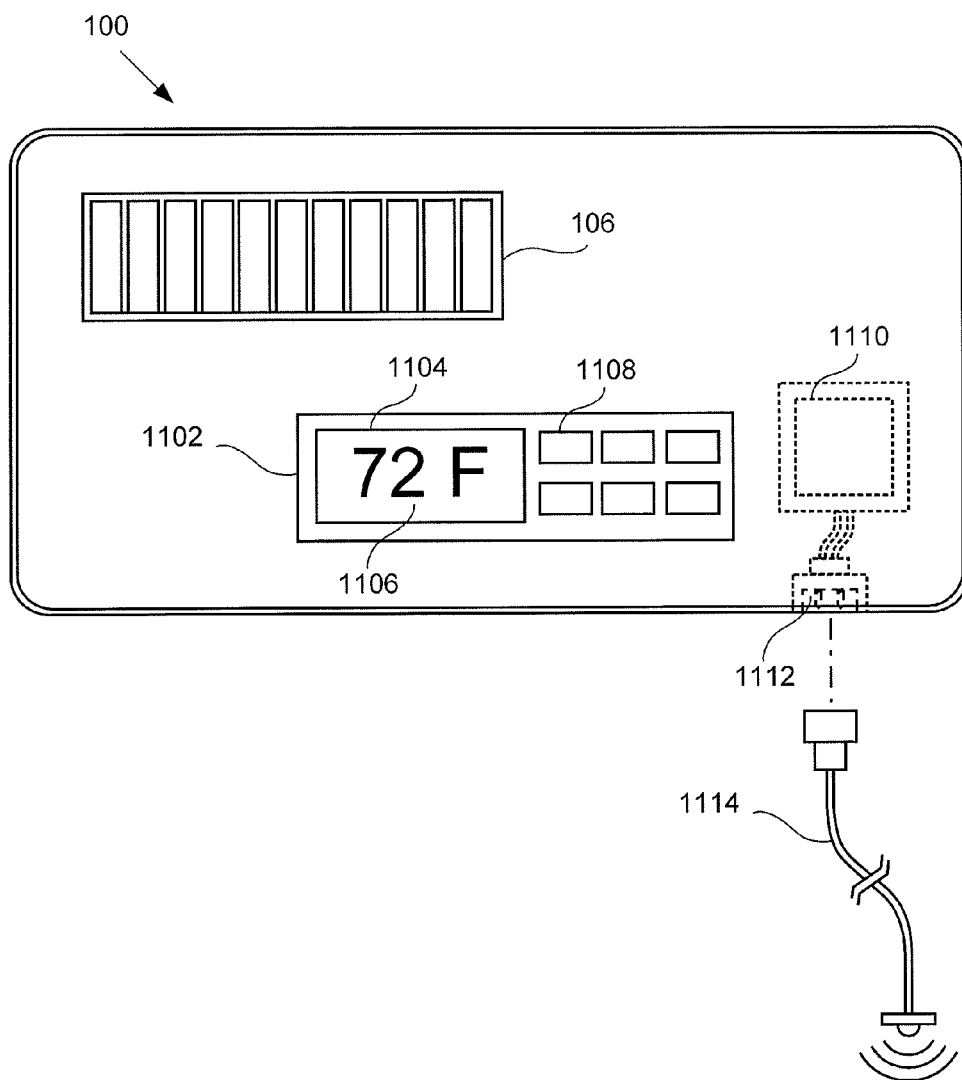


FIG. 11

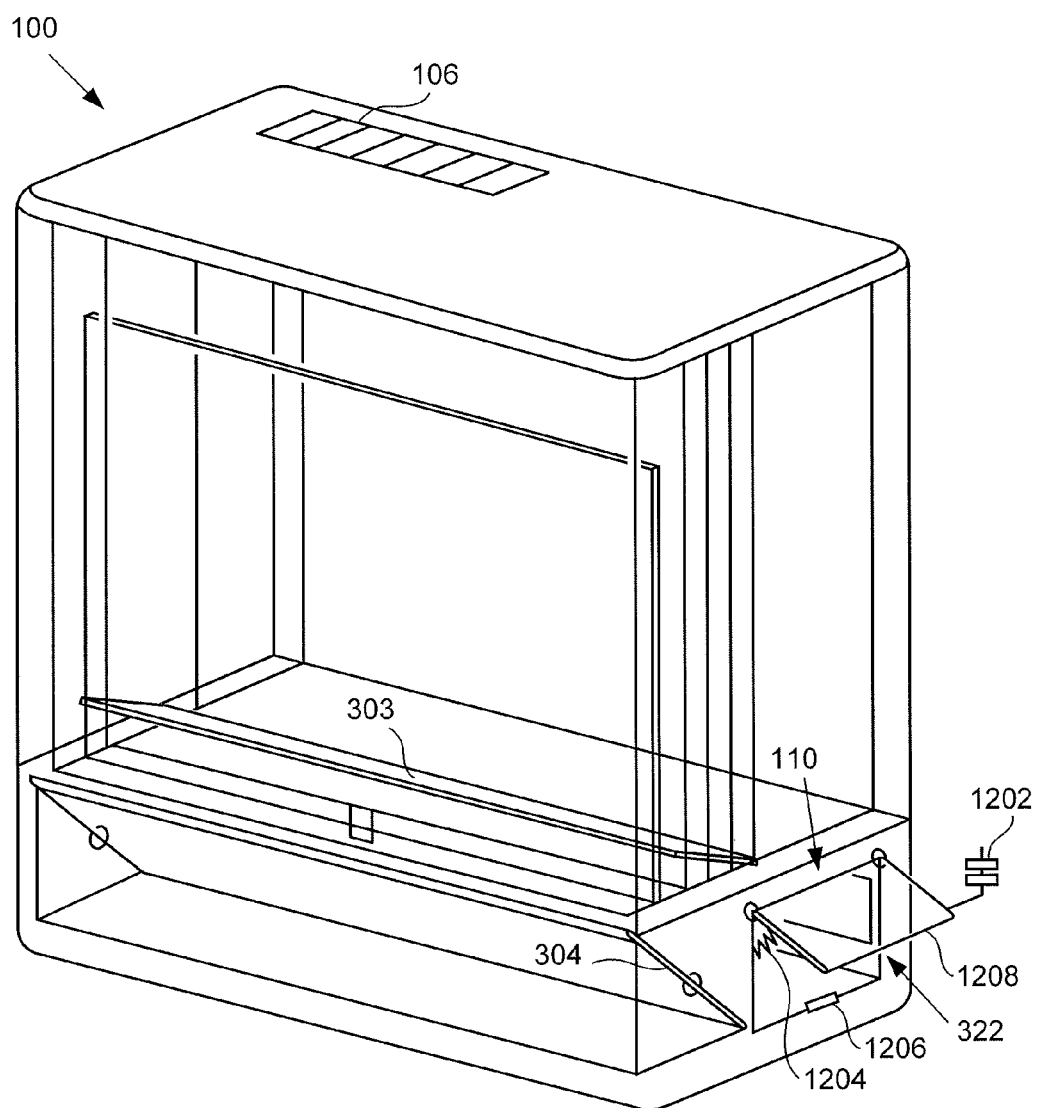


FIG. 12



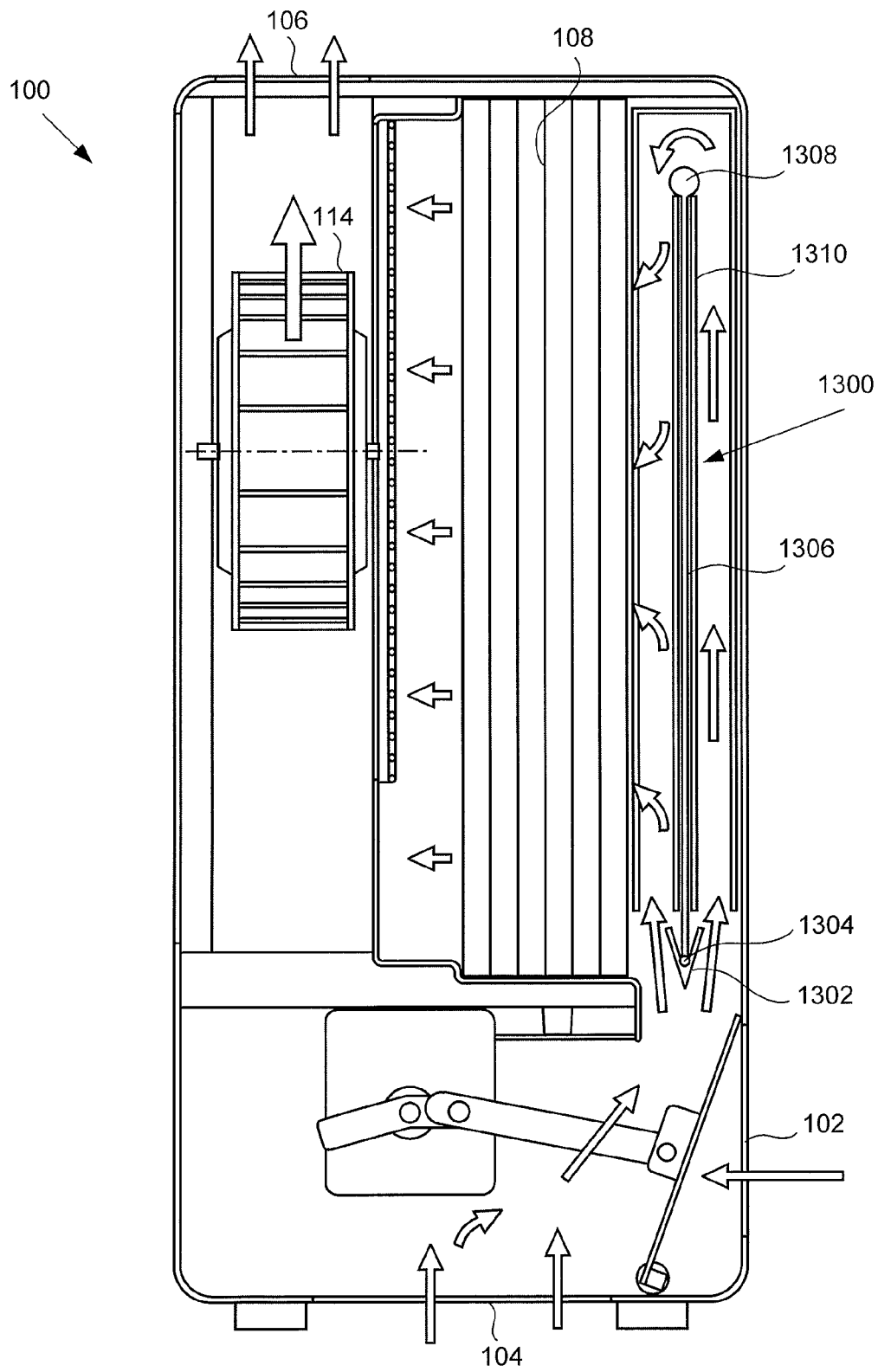


FIG. 13

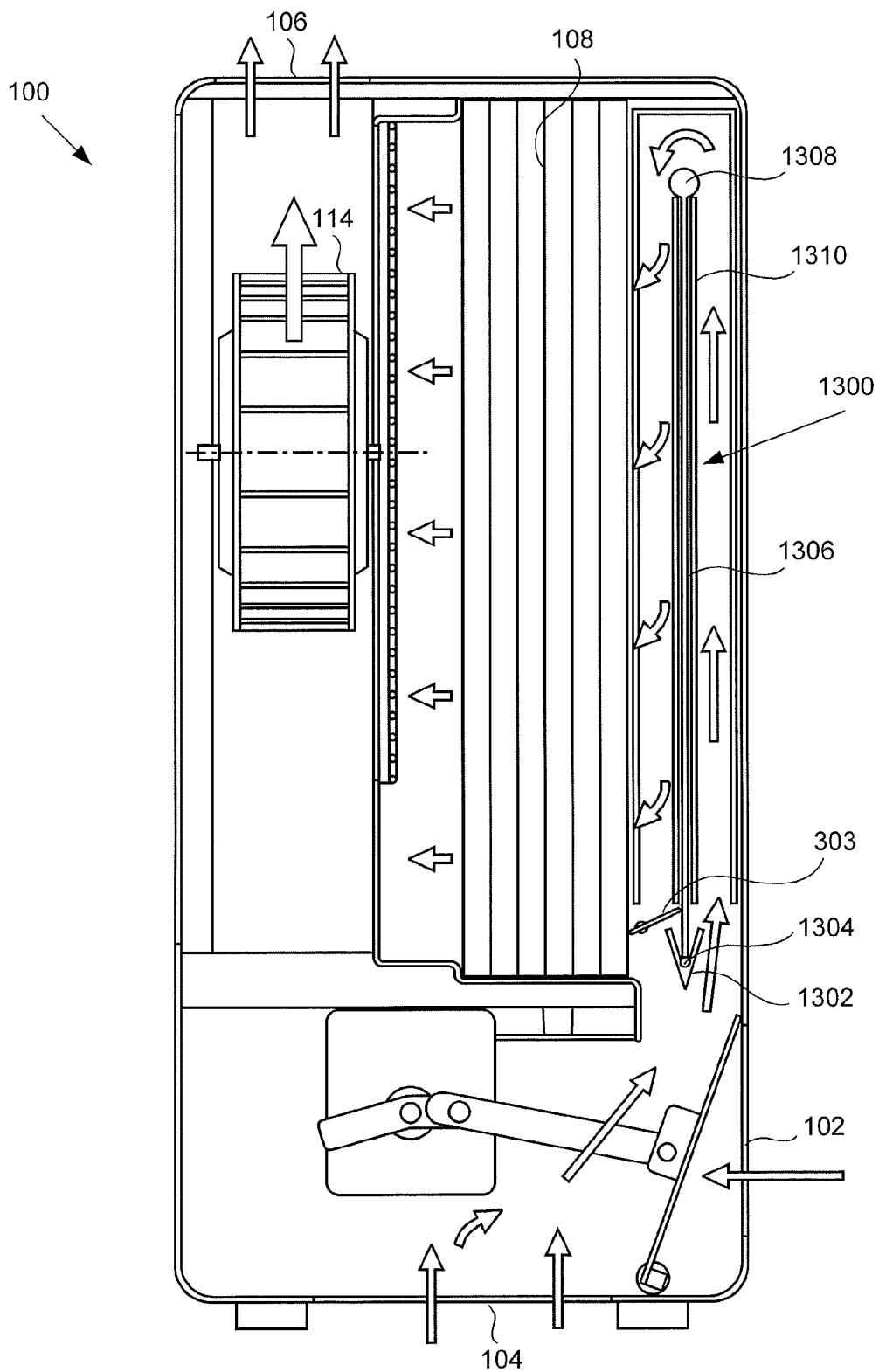


FIG. 14

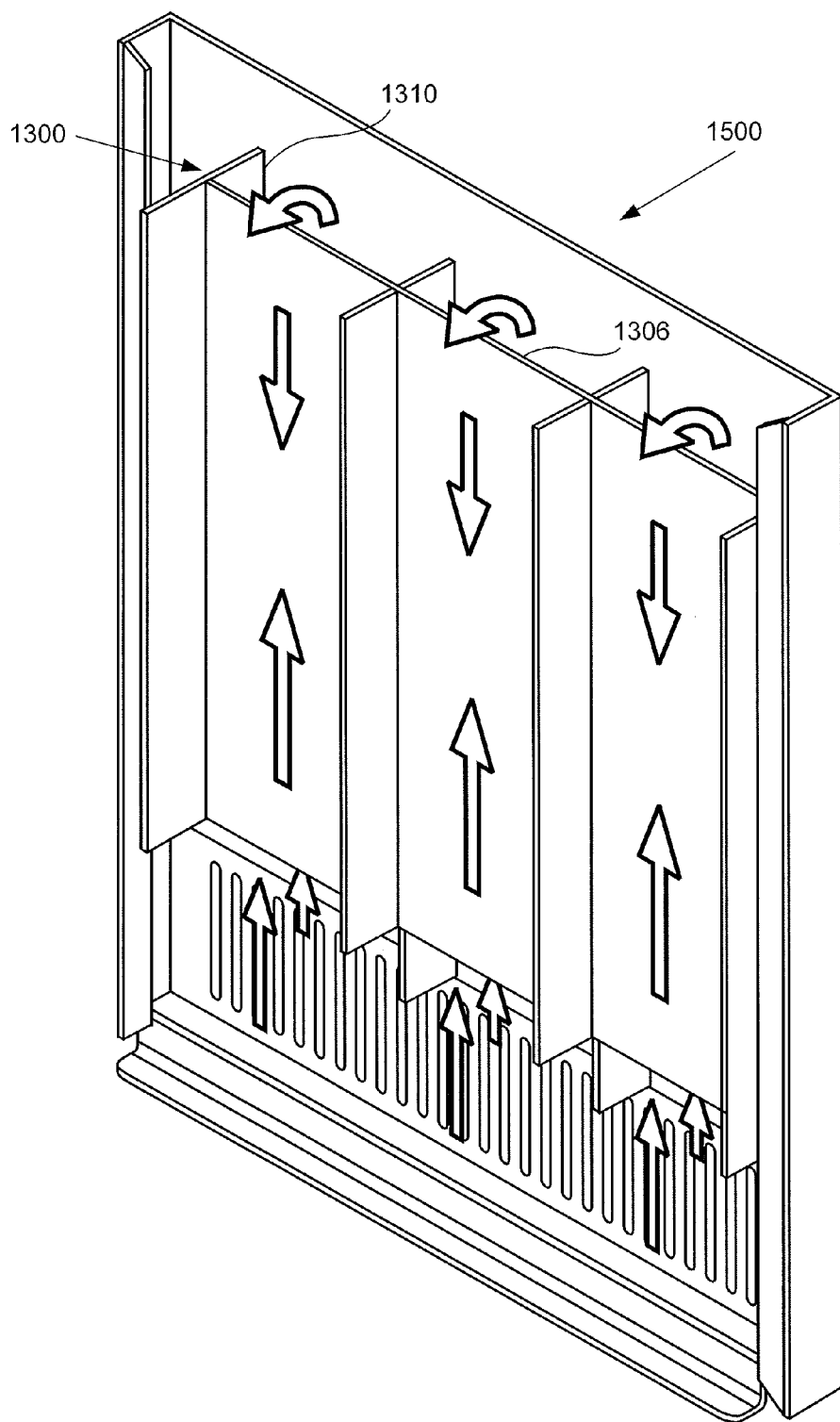


FIG. 15

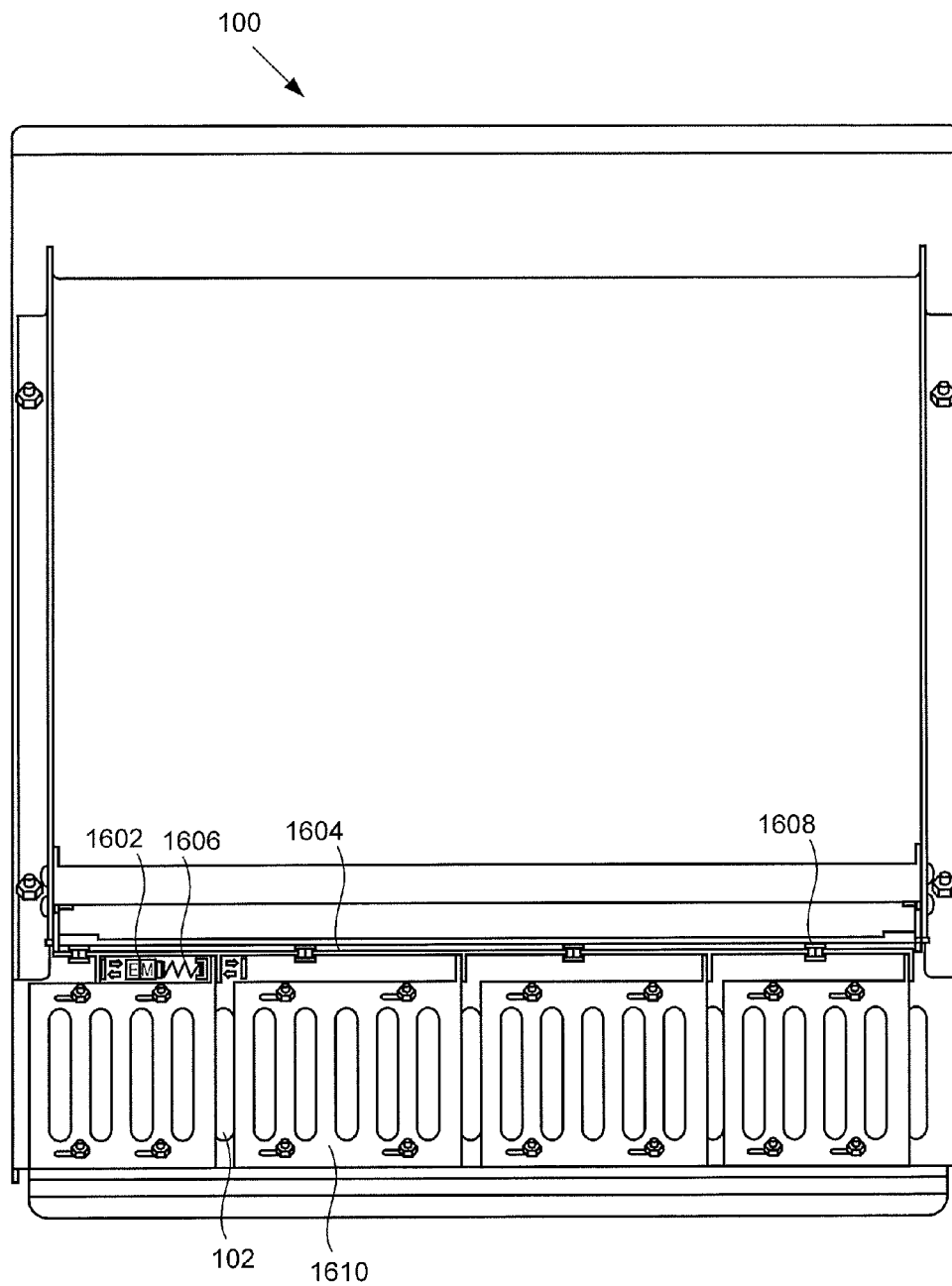


FIG. 16

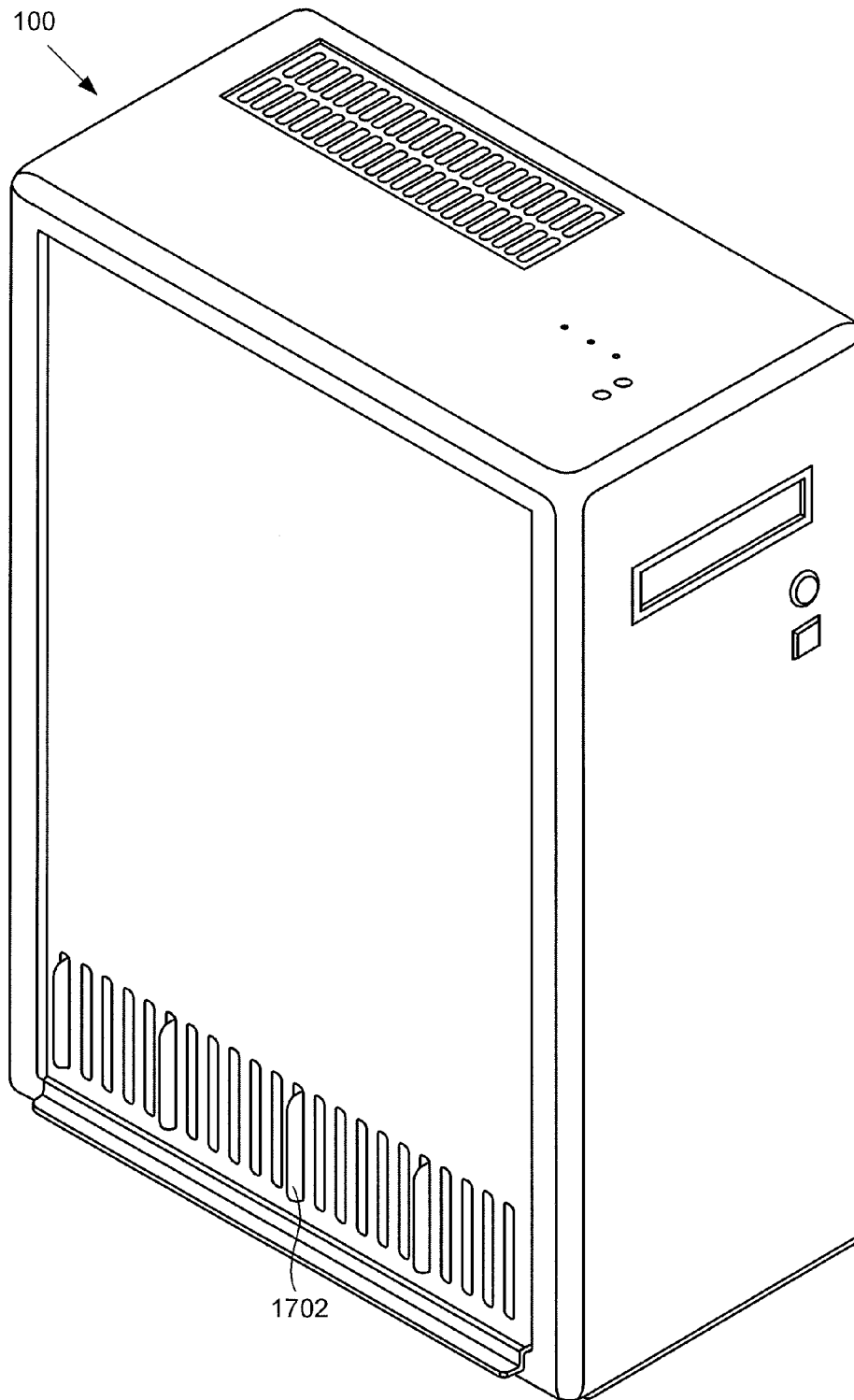


FIG. 17

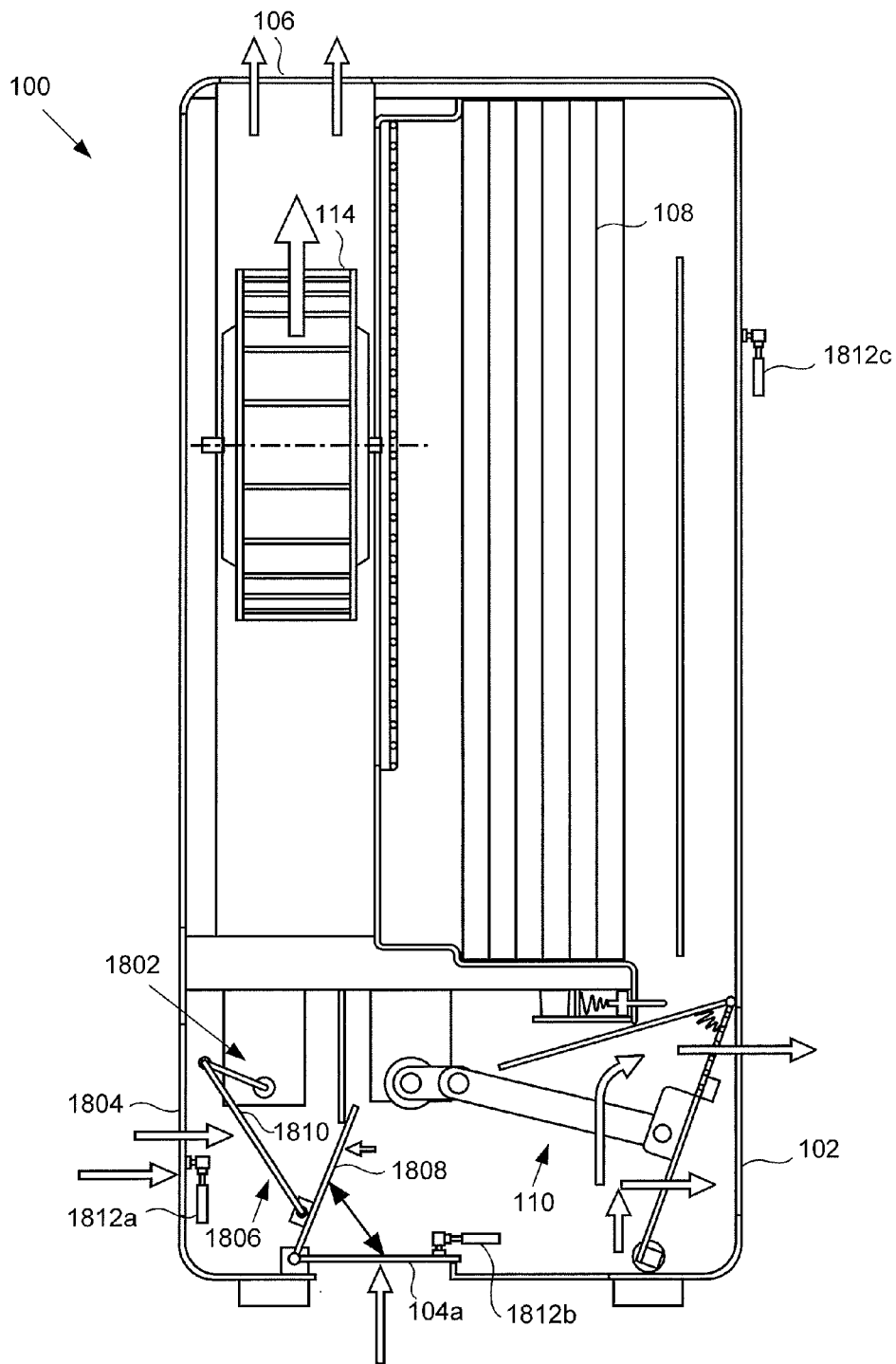


FIG. 18

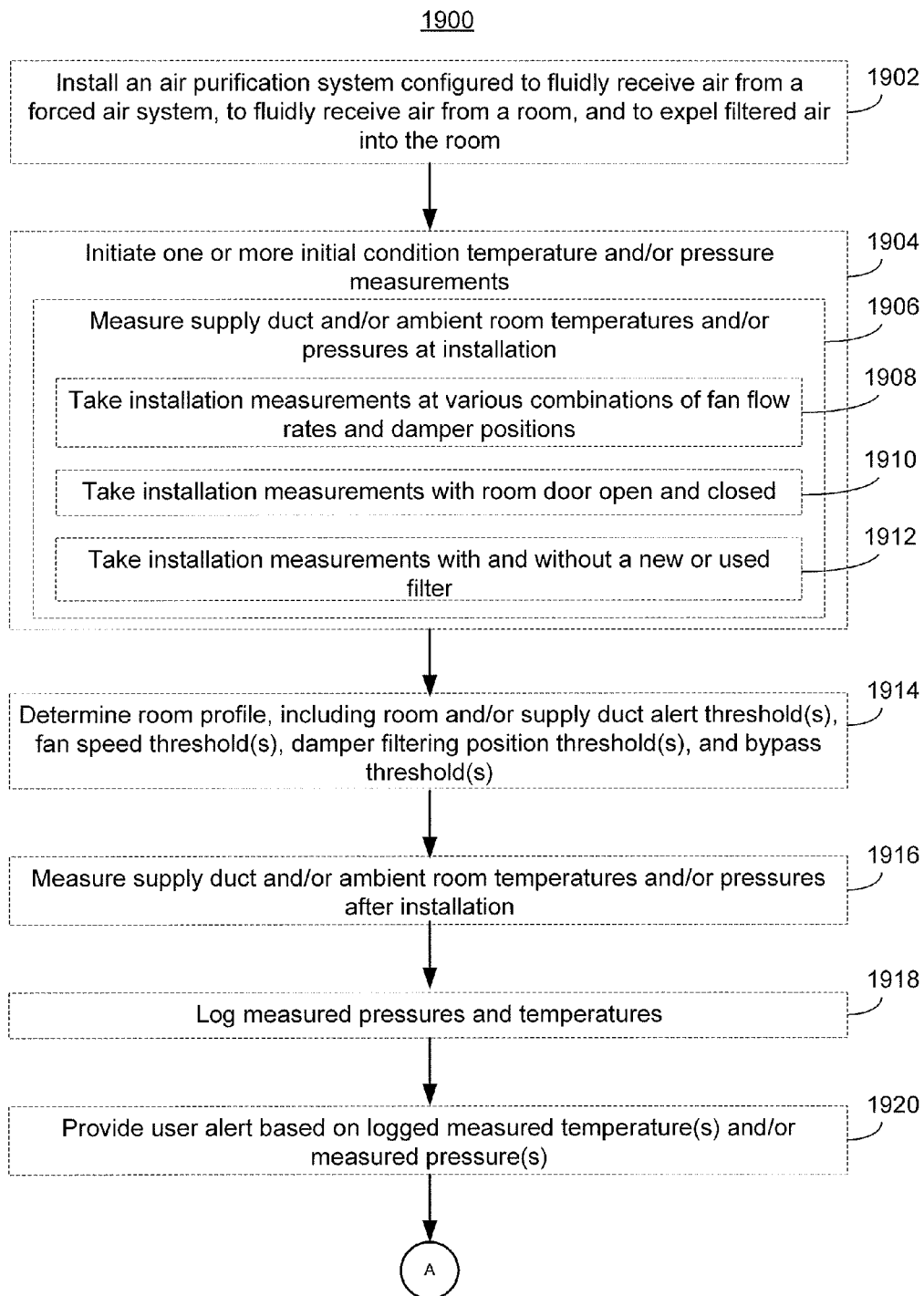


FIG. 19A

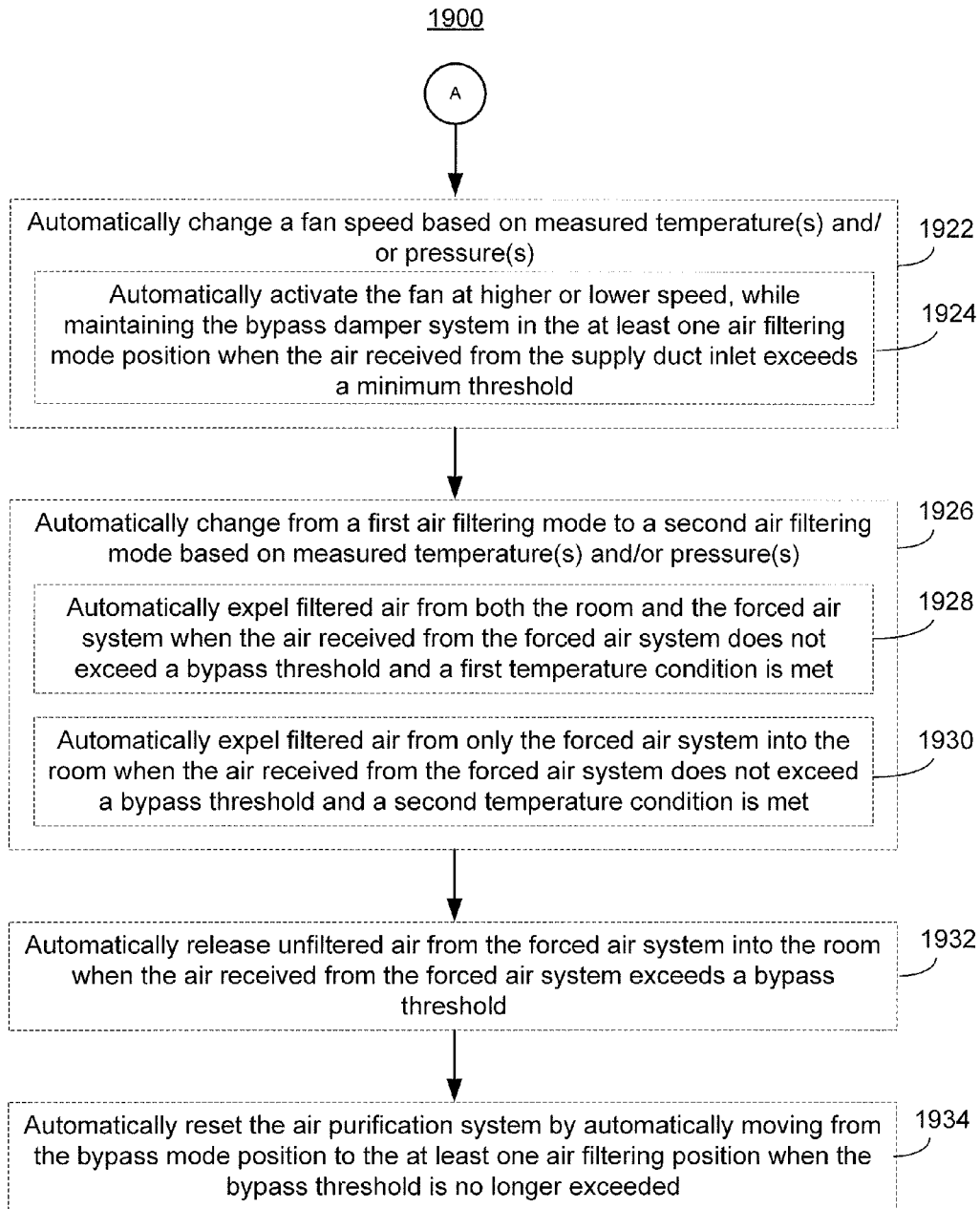


FIG. 19B



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**ROOM AIR PURIFIER WITH  
PRESSURIZATION RELIEF**

## TECHNICAL FIELD

The disclosed embodiments relate generally to purification of air and, more specifically to provide a room air purifier with pressurization relief.

## BACKGROUND

Indoor air quality ranks among the top five environmental risks to humans in the United States. The costs are measured in tens of millions of people suffering from allergies and asthma and more formidable airborne VOCs, gases, and other contaminants. The cost to the United States is tens of billions of dollars in health care and lost productivity at work. For instance, the EPA has an exhaustive list of the consequences of poor indoor air quality. As such, it is desirable to provide solutions to indoor air quality.

## SUMMARY

Treating air provided from forced air heating and/or cooling systems, which are commonly found in roughly 80% of the homes in the United States (per US Census data) is one way to address improving the above described indoor air quality issues. Furthermore, when dealing with forced air systems it is also desirable to protect the forced air system with pressurization relief capabilities.

The air purification system described in this application is suited to control the source of contaminated air before it enters a room. The air purification system cleans the air before it can enter the room, and re-cleans the cleaned room air. In some embodiments, it also prevents air from outside of the room from entering the cleaned room through a doorway. Furthermore, in some embodiments, the air purification system also serves as a room zone and periodically takes steps to control room temperature. The air purification system also provides a, supply duct over pressure relief capability. Other benefits of some embodiments include user notification/alert and corrective action utilizing the system's internal components. In addition, the air purification system monitors supply duct air temperature and can notify the user of the forced air systems efficiency and performance. In some implementations, room pressure is also monitored to alert the user and to automatically manage its internal systems to insure an appropriate amount of air is returned to the forced air system. Other features allow the air purifier to be turned "off" or unplugged while still installed and means are provided to allow air from a forced air supply duct to flow freely into the room without being drawn through the filter or even coming into contact with the filters. In this manner filter life can be extended and the filter only used when the unit is operating. Additionally, any contaminants on the face of the filter are prevented from mixing with supply air and are prevented from entering into the room in unhealthy concentrations. A forced air system's supply air can continue to heat, cool, or ventilate the room with minimal interference from the non-operating air purifier unit. Communications capabilities of some embodiments allows the air purification system to communicate with other air purification systems and to coordinate controls to alleviate undesirable supply duct pressure and room pressure while continuing to optimize the room temperature in multiple rooms where they are installed. In some embodiments, communication with other simpler systems is also possible. For example, one air purification system with a full complement

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of accessories can manage and take responsibility for corrective action for other systems with fewer features and capabilities and will, in some embodiments, provide important operating instructions to the other purifier systems.

The following presents a summary of the invention in order to provide a basic understanding of some of the aspects of the invention. This summary is not an extensive overview of the invention. It is not intended to identify key/critical elements of the invention or to delineate the scope of the invention. Its sole purpose is to present some of the concepts of the invention in a simplified form as a prelude to the more detailed description that is presented later.

Various embodiments of systems, methods and devices within the scope of the appended claims each have several aspects, no single one of which is solely responsible for the desirable attributes described herein. Without limiting the scope of the appended claims, some prominent features are described herein. After considering this discussion, and particularly after reading the section entitled "Detailed Description" one will understand how the features of various embodiments are used.

One aspect of the disclosure is an air purification system with over pressurization relief capabilities including the following components. A room air inlet is configured to fluidly receive air from a room. A supply duct inlet is configured to fluidly receive air from a forced air system. An air filter/purifier is configured to filter/purify air. An outlet is configured to expel filtered/purified air into the room. A bypass damper system has at least one air filtering mode position and a bypass mode position. A pressure control mechanism is configured to automatically allow the bypass damper system to move from at least one air filtering mode position to a bypass mode position when the air received from the supply duct inlet exceeds a bypass threshold. Furthermore, at least one air filtering mode position guides air from the supply duct inlet and air from the room air inlet to both pass through the filter and the outlet, and impedes air from passing from the supply duct inlet to the room air inlet. Also, the bypass mode position guides air from the supply duct inlet into the room without requiring passage through the filter and the outlet.

Another aspect of the disclosure is a method of providing pressurization relief to an air purification system. An air purification system configured to fluidly receive air from a forced air system, to fluidly receive air from a room, and to expel filtered/purified air into the room is provided. Air pressure from the forced air system is measured. Room air temperature is automatically measured. Unfiltered air from the forced air system is automatically released into the room only when the air received from the forced air system exceeds a bypass threshold. Filtered/purified air from both the room and the forced air system is automatically expelled when the air received from the forced air system does not exceed a bypass threshold and a first temperature condition is met. Furthermore, filtered/purified air from only the forced air system is automatically expelled into the room when the air received from the forced air system does not exceed a bypass threshold and a second temperature condition is met.

Thus, these systems and methods provide new and efficient ways to provide clean, filtered/purified air directly from a forced air system before it reaches the room (with or without a mixing the air from the forced air system with already filtered/purified room air depending on various conditions.) These systems and methods also include new and efficient ways to automatically provide over pressurization relief capabilities to ensure that the forced air system is not harmed by the air purification system.

## BRIEF DESCRIPTION OF THE DRAWINGS

For a better understanding of the aforementioned aspects of the invention as well as additional aspects and embodiments thereof, reference should be made to the Description of Embodiments below, in conjunction with the following drawings in which like reference numerals refer to corresponding parts throughout the figures.

FIG. 1 is a side section view of an air purification system in accordance with one embodiment of the present invention, wherein the bypass damper system is in a bypass mode position.

FIGS. 2A and 2B are side section views of the embodiment of FIG. 1, wherein the bypass damper system is in two distinct air filtering/purifying mode positions.

FIG. 2C is a perspective view of the bypass damper system of the embodiment of FIG. 1.

FIG. 3 is a side section view of an air purification system in accordance with another embodiment of the present invention, wherein the bypass damper system is in a bypass mode position.

FIG. 4 is a side section view of the embodiment of FIG. 3, wherein the bypass damper system is in an air filtering/purifying mode position.

FIG. 5 is a rear section view of the embodiment of FIG. 3, wherein the bypass damper system is in an air filtering/purifying mode position.

FIG. 6 is a rear section view of the embodiment of FIG. 3, wherein the bypass damper system is in a bypass mode position.

FIG. 7 is a rear section view an air purification system in accordance with another embodiment, wherein the bypass damper system is in a bypass mode position.

FIG. 8 is a rear section view an air purification system in accordance with yet another embodiment, wherein the bypass damper system is in a bypass mode position.

FIG. 9 is a side section view of a gravity bypass damper system.

FIGS. 10A-10F illustrate the components of the air purification system that provide convenient filter disposal.

FIG. 11 is a top view of an air purification system in accordance with some embodiments.

FIG. 12 is a perspective view of an air purification system in accordance with still another embodiment.

FIG. 13 is a side section view of an enhanced air distribution channel for the air purification system.

FIG. 14 is a side section view of an embodiment similar to the embodiment of FIG. 13 with the addition of a check damper.

FIG. 15 is a perspective view of the air purification system's front panel.

FIG. 16 is an internal side view of an air purification system pressure control mechanism.

FIG. 17 is a perspective view of the outside of an air purification system with a manual reset option for the pressure control mechanism.

FIG. 18 is a side section view of another embodiment of an air purification system.

FIGS. 19A-19B are flowcharts illustrating a methods of providing pressure relief and alerts and action in accordance with some embodiments.

Like reference numerals refer to corresponding parts throughout the drawings.

## DESCRIPTION OF EMBODIMENTS

Reference will now be made in detail to embodiments, examples of which are illustrated in the accompanying draw-

ings. In the following detailed description, numerous specific details are set forth in order to provide a thorough understanding of the present embodiments. However, it will be apparent to one of ordinary skill in the art that the present various embodiments may be practiced without these specific details. In other instances, well-known methods, procedures, components, and networks have not been described in detail so as not to unnecessarily obscure aspects of the embodiments.

It will also be understood that, although the terms first, second, etc. may be used herein to describe various elements, these elements should not be limited by these terms. These terms are only used to distinguish one element from another. For example, a first element could be termed a second element, and, similarly, a second element could be termed a first element, without changing the meaning of the description, so long as all occurrences of the first element are renamed consistently and all occurrences of the second element are renamed consistently. The first element and the second element are both elements, but they are not the same element.

The terminology used in the description of the embodiments herein is for the purpose of describing particular embodiments only and is not intended to be limiting of the claims. As used in the description of the embodiments and the appended claims, the singular forms "a," "an," and "the" are intended to include the plural forms as well, unless the context clearly indicates otherwise. It will also be understood that the term "and/or" as used herein refers to and encompasses any and all possible combinations of one or more of the associated listed items. It will be further understood that the terms "comprises" and/or "comprising," as well as the terms "includes" and/or "including" when used in this specification, specify the presence of stated features, integers, steps, operations, elements, and/or components, but do not preclude the presence or addition of one or more other features, integers, steps, operations, elements, components, and/or groups thereof.

As used herein, the term "if" may be construed to mean "when" or "upon" or "in response to," depending on the context. Similarly, the phrase "if it is determined" or "if (a stated condition or event) is detected" may be construed to mean "upon determining" or "in response to determining" or "upon detecting (the stated condition or event)" or "in response to detecting (the stated condition or event)," depending on the context.

In some embodiments, when the an air purification system is used with homes equipped with forced air heating and or cooling systems and the associated ductwork or other high velocity piped delivery systems, the room to receive air filtering/purifying treatment may have multiple supply ducts and registers. Air filtering can be accomplished by installing the air purification system on only one supply register. In such instances, others registers are blocked to provide the most efficient/effective room air filtering/purification. The result is, typically, more air will come out of or be drawn out of the remaining open duct than would have been the case if additional ducts were still open. Furthermore, some homes have multiple forced air delivery zones, for example, upstairs and downstairs. The upstairs zone may have fewer rooms served and may also have fewer supply registers. In these cases, the upstairs registers may move more air, per register, since the air handler/blower, in the forced air unit may only operate at one speed in most installations. Blocking the other supply ducts in the one room or even two rooms will have little impact on some forced air systems (e.g. forced air systems are typically sized large enough to handle a higher volume of air through a reduced number of ducts). However, if enough ducts are blocked and air purifier systems are installed on all

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or many of the remaining open ducts, in some instances the forced air system may become over pressurized. In other words, the pressure at the air purification system could reach a level deemed to have a negative impact on the forced air system. For instance, the pressure may reach an undesirable level of 0.2 inches of water column, for one example, for a particular forced air system installation.

As such, it is advantageous to provide a mechanism and method for providing pressure relief in order to overcome various issues such as those described above. Specifically, it is advantageous to provide an air purification system with over pressurization relief capabilities as described herein.

The air purification system with over pressurization relief capabilities described herein addresses over-pressurization relief for both individual rooms and for an entire home. For instance, an air purification system can be mounted in every room of a user's home without fear of causing over pressurization to the home's forced air system. The air purification system(s) are designed to manage and adapt to the variables found in forced air systems worldwide. As such, new functional requirements are described in the following figures and paragraphs to manage the spectrum of forced air system installations, and the homeowners who operate them. This air purification system also has applications in commercial buildings.

FIG. 1 is a side view of an air purification system with over pressurization relief capabilities 100. In this embodiment the air purification system 100 is configured for sitting on top of a supply register, sealing onto it, and cleaning the air before it enters the room. It should be understood that the air purification system 100 can be connected to any supply register whether on the floor, the wall, or the ceiling without departing from the functions described with respect to this exemplary embodiment. The air purification system 100 includes a room air inlet 102 configured to fluidly receive air from a room, a supply duct inlet 104 configured to fluidly receive air from a forced air system, and an outlet 106 configured to expel filtered/purified air into the room as shown. It is noted that FIG. 1 illustrates a bottom located supply duct inlet 104a for receiving air from a floor vent, but supply duct inlets are located wherever is necessary to receive supply air such as the optional side located supply duct inlet 104b located at the back of the air purification system 100 configured to receive supply air from a wall vent or ceiling vent. In some embodiments, one or more of the supply duct inlets 104 are configured to receive air from a source other than the force air system (e.g., through a wall from another room such as a heated great room). However, the embodiments described in detail below assume a supply inlet 104 receives air from a force air system.

Other components of the air purification system 100 include a housing 112, a bypass damper system 110 in fluid communication with the supply duct inlet 104 and the room air inlet 102, one or more filters (or similar, or other air purification means) 108 configured to filter or purify air, and a fan 114 to draw air into the air purification system 100, through the filters 108 and then expel cleaned air into the room through the outlet 106. It is noted that in some embodiments the fan 114 is an adjustable speed fan having at least a high speed and a low speed.

As will be explained in more detail with respect to FIG. 15, in some embodiments, one or more air distribution panels 116 are included to evenly distribute air over the face of the filters 108 to improve the distribution of air for contaminate collection, to equally slow and balance the flow of air for maximum filtration and absorption, and/or to increase the flow of air by

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distributing it evenly over the entire area of the filters (or similar air purification means) 108.

FIG. 1 also shows one or more optional ionizers 118. The ionizers 118 release ions which cause positively charged contaminants to clump together and grow in size for improved collection by filters 108. In some embodiments, an ionizer(s) 118a is positioned at or near the supply duct inlet 104. This supply duct ionizer 118a creates an opportunity to concentrate the ionization treatment of the dirtiest air (the air received from the forced air system). In some embodiments, an ionizer 118b is additionally or alternatively positioned in the blend chamber 120. The blend chamber 120 is a space within the air purification system 100 where air received from the supply duct inlet 104 and air received from the room air inlet 102 is allowed to blend together prior to passing through the one or more filters 108. Specifically, in FIG. 1 the blend chamber 120 is located between the supply duct inlet 104, the room air inlet 102, the bypass damper system 110, and the air distribution panels 116. The blend chamber 120 is configured to receive room air and supply duct air independently and simultaneously depending on the positioning of the bypass damper system 110 as will be explained in more detail below. The blend chamber ionizer 118b creates an opportunity for ionization treatment to be evenly applicable to all air prior to entering the filters 108 regardless of the positioning of the bypass damper system 110. In some embodiments, an ionizer 118 is also provided at an air outlet 106 in order provide ionized air to the room, as some consumers appreciate health benefits associated with negatively ionized air. For instance, some users believe that ionized air manages the hormone level in the body which helps in preventing seasonal affective disorder disease which results into depression. Negative air ionizers can also be useful to restore balance to the number of positive ions in a room, often generated by many electrical appliances like televisions and computers.

FIG. 1 illustrates a bypass damper system 110 which includes an air inlet damper 122. The air inlet damper 122 includes a two blade damper mechanism having a first blade 124 and a second blade 126. The first blade 124 defines a plurality of through holes (or slots) 128 configured to allow air to pass there through (as is shown in more detail in FIG. 2C). The second blade 126 is configured to cover and uncover the through holes (or slots) 128 in the first blade 124.

As illustrated in FIGS. 1, 2A, 2B, and 2C the pressurization relief capabilities of the air purification system 100 utilizing a bypass damper system 110 which includes an air inlet damper 122 having a two blade damper mechanism functions as follows. The bypass damper system 110 includes one or more air filtering/purifying mode positions (as illustrated in FIGS. 2 and 3) and a bypass mode position (as illustrated in FIG. 1).

As illustrated in FIG. 1, in the bypass mode position, the first blade 124 is in a first position, covering but not blocking the room air inlet 102 because it's through holes (or slots) 128 are exposed. The second blade 126 is in a second position, which blocks most or all of the air received from the supply duct inlet 104 from entering the blend chamber 120 and subsequent filters 108, and at the same time uncovers the through holes (or slots) 128 in the first blade 124. As such, when the bypass damper system 110 with an air inlet damper 122 is in the bypass mode position (as illustrated in FIG. 1), the air inlet damper 122 guides air from the supply duct inlet 104 into the room without requiring that the air pass through the filter(s) 108 and outlet 106. As such, the air from the forced air system 100 can safely pass from the supply duct inlet 104 pass into the room through the room air inlet 102. In some embodiments, a spring 134 holds the second blade 126 in the bypass mode position. In some embodiments, the sec-

ond blade is positioned to block and seal out supply duct air from reaching the filters **108**, while simultaneously opening a channel for supply duct air to exit the unit through the room air inlet. One advantage of blocking the supply duct air, from reaching the filters **108** is that the air thus will not contact contaminants on the face of the filter and risk placing these contaminants into the room. Another advantage is the filters **108** are not used, prolonging their life, when the air purification system is not running. As such, the placement of the first blade **124** and the second blade **126** of the air inlet damper **122** in the bypass mode position provides both overpressurization relief and contamination safety. In some embodiments, a check damper is also provided to further ensure that the air release through the room air inlet has not contacted contaminants on the face of the filter (e.g., a check damper **303** is illustrated in FIG. 3.)

Thus, this bypass mode position—the combined positioning of the first blade **124** in the first position and the second blade **126** in the second position—is configured to relieve supply duct pressure. It is noted that the bypass mode position is also utilized in situations when the supply duct pressure need not be relieved. For instance, in some embodiments the bypass mode position is the default “off” position for the air purification system **100**. In some embodiments, the air purification system **100** is configured to automatically move to the bypass mode position when the air purification system **100** is turned off, unplugged, or malfunctions. As such the air from the forced air system can safely and easily pass from the supply duct inlet **104** out into the room through the holes (or slots) **128** in the first blade **124** covering the room air inlet **102**. One advantage of allowing air to pass into the room through the holes **128** in first blade **124** while it covers the room air inlet **102** rather than completely uncovering the room air inlet **102** is that the first blade **124** protects the air purification system **100** from being exposed. For instance the first blade **124** keeps pets and children from touching the interior mechanism of the air purification system **100** and potentially breaking the mechanisms or injuring themselves, without additional protective grills. This protective position is thus especially useful to be the default and off position for the air purification system **100**.

The air purification system **100** also includes a pressure control mechanism **129**. In some embodiments, as shown in FIG. 1, the pressure control mechanism **129** includes a pressure sensor **130**, which is configured to measure the pressure of the air received from the supply duct inlet **104**, and an activation means such as an electronic controller **132**. The pressure control mechanism **129** is configured to automatically allow the bypass damper system **110** to move from an air filtering/purifying position to a bypass mode position when air received from the supply duct inlet exceeds a bypass threshold. In some embodiments, the bypass damper system **110** is specifically controlled by a motor **148**, which controls the first blade **124** that moves from the first position (covering the room air inlet **102**) to the third position (uncovering the room air inlet) and various positions in-between. Furthermore, in some embodiments, movement of the second blade **126** from the second position (covering the holes **128** in first blade **124**) and the fourth position (uncovering the holes **128** in the first blade **124**) is controlled by the electronic controller **132** in conjunction with an electromagnet **136** and spring **134** as explained below. In some other embodiments, as shown in FIG. 12, pressure control is being accomplished through weighted gravity by pass dampers. In other embodiments, the pressure control mechanism **129** is configured to have the electronic controller **132** automatically allow the bypass damper system **110** to move from an air filtering/purifying

mode position to a bypass mode position when the measurement of the pressure sensor **130** exceeds a bypass pressure threshold. In some embodiments, a supply duct pressure is automatically initially read during/after installation of the unit. Then a threshold value is added to the initial reading, and this is automatically becomes the bypass threshold. In some embodiments, the bypass pressure threshold depends on the forced air system and the air handler including the size and length of the ducts and register. In some embodiments, the bypass pressure threshold is determined by utilizing a “fixed set point” sensor, a technician setting the bypass threshold, or a combination of both. In some embodiments, more than one bypass threshold is set. For instance, there may be an “open door” bypass pressure threshold and a “closed door” bypass pressure threshold (when the room is closed/sealed from adjoining rooms). Furthermore there may be a “new” filter bypass pressure threshold distinct from a “used” filter bypass threshold, which may be a sliding amount re-adjusted on a periodic based (e.g., weekly, monthly, bi-yearly, or yearly). In the settable option, a HVAC technician assesses the system and sets a high limit. In some instances adequate instructions are provided so that consumers can perform the HVAC technician actions and set a bypass threshold themselves. A consumer or technician may read the digital pressure reading with only one unit operating in 1 room and place the set point above that point. For instance, it is useful to know the forced air systems normal duct pressure when setting the set point.

In some preferred embodiments, the air purification system automatically reads (and optionally records) the supply duct pressure utilizing the pressure sensor **130** and adds a predetermined value to it (plus 0.05 to 0.1 inches of water column, for example). In some embodiments, this is considered the home’s normal operating range. In this instance, a maximum bypass pressure threshold is established without human input. It is noted that the measurement taken by the pressure sensor **130** with a new furnace filter in place can also be recorded and used as a maximum set point for a decreased supply duct pressure.

Furthermore, when additional purifier units are added to other rooms, they can adopt the same bypass pressure threshold (plus 0.05 to 0.1 inches of water column, for example). As such, the additional air purification units need not be equipped with mechanisms for automatically measuring supply duct pressure and independently setting an operating range. In some embodiments the original (more advanced) air purification unit is also equipped with mechanisms to control the additional (less advanced) air purification units to take relief actions described in this application. Furthermore, in some embodiments, the more advanced air purification unit provides on its display screen suggestions for customer action regarding the other unit(s) such as suggesting opening a room door, replacing a filter, or even opening supply registers in other (e.g., unused) rooms to restore peak air flow and efficiency to the forced air system. As such, the consumer can save money on utilities. Furthermore, low forced air system air flow, can provide the user with a warning of forced air system failures yet to be fully recognized.

In some embodiments, the electronic controller **132** is configured to relieve supply duct pressure by first increasing the speed of the fan **114** prior to activating the bypass damper system to move from an air filtering mode position to a bypass mode position. For instance, the electronic controller **132** is configured to activate the fan at a high speed, while maintaining the bypass damper system **110** in the at least one air filtering/purifying mode position (e.g. FIG. 2A or FIG. 2B) when the air received from the supply duct inlet **104** exceeds a minimum threshold. It is noted that this minimum threshold

is lower than the bypass threshold required by the electronic controller to activate the bypass damper system to move from an air filtering/purifying mode position to a bypass mode position. For instance, in some embodiments the minimum threshold is half of the bypass threshold.

An automatic disengagement of the first blade **124** and the second blade **126** takes place as follows. When supply duct air pressure exceeds a threshold (e.g., as measured by the pressure sensor **130**) or if the unit is turned “off”, is unplugged, or malfunctions the electromagnet **136** will be caused to lose power and the second blade **126** will be released and the first blade **124** will return to the first position covering the room air inlet **102**. As such, air from the supply duct inlet **104** can exit directly into the room as illustrated in FIG. 1. In other words, if the unit is off for any reason the first blade **124** and the second blade **126** will assume the “safety position” illustrated in FIG. 1. It is noted that the supply duct air is prevented from mixing with contaminants on the face of the filters **108** because of the blocking position of the second blade **126**.

Automatic re-set joining (magnetically latching) of the first blade **124** and the second blade **126** is completed as follows. In some embodiments, the joining is performed each time the air purification system **100** starts or restarts. The first blade **124** moves to the third position and tab **138** pushes the second blade **126** into position to align and seal against the first blade and engaged/held in place by the electromagnet **136**. In some other embodiments, the dampers are re-joined manually.

In some embodiments, the pressure sensor **130**, located in the supply duct air entry area automatically measures pressure which is reflective of supply duct pressure and overpressure. In other embodiments the pressure sensor **130** also includes a solenoid valve **140** that enables the pressure sensor **130** to collect pressure data from either the supply duct air **142** or from the ambient room environment **144**. In some embodiments, the pressure sensor is an MPL 9371 model or 9391 model, which is appropriate for this dual purposing. A custom MPL 3115A2 may also be adapted for the low pressure sensing and data collection. In some embodiments, more than one pressure sensor is used instead of the dual purpose sensor. In some embodiments, the pressure sensor **130** has a digital display/output of pressure values which can be configured to be multiple set points for pressure thresholds stored and to be acted as described herein. In some embodiments, the Solenoid Valve **140** is a 3 way or 4 way center closed configuration for simplicity such as a model SCO726G by ONBO Electronic Co. LTD. A solenoid valve **140** is useful for sampling supply duct air pressure **142** and ambient air pressure **144** alternately, and continuously and most specifically once a high pressure limit is exceeded to monitor corrective action.

For instance, the air purification system **100** can be used to create a positive pressure in a room and therefore positive ventilation of the room to keep the room completely full of clean and filtered air, within a maximum room pressure threshold. In some embodiments, this room pressure threshold can be determined during the room air purifier installation with the customer following prompts to open and close the room door(s) while the air purification system unit runs combinations of fan flow rates and damper positions, logs and interprets the data to establish the maximum room pressure threshold value. Other prompts may include removing the forced air systems filter and providing time for the purifier system to run an installation program, collect a range of data for interpretation and future action. In some embodiments, other prompts will ask that a new filter be installed so the same installation program data can be collected for comparison to the no filter data. It is further noted that in some embodiments flow measurements are adjusted to account for the air density

effects caused by temperature and altitude. For instance, the initial pressure data collected is normalized to a range of standard temperatures in the supply duct.

In some implementations a prompt for a used, ready to be discarded, filter is requested to be temporarily installed or re installed so comparative data can be collected to help establish filter condition and future filter change notifications/alerts.

Even without prompts, some embodiments of the air purification system will automatically log data over time. This data includes any of the following or a combination thereof: supply duct air pressure and temperature, ambient room temperature data versus room temperature set point, fan speeds, damper positions and fan speeds, and ambient room air pressure. The data is logged and interpreted over time to pinpoint room door open and door closed events and when they occur. Thus the room air purification system can run performance sequences of fan speed and damper positions and thereby automatically establish a maximum room pressure threshold without consumer assistance. A room profile is thereby created by the air purification system. In some embodiments, ambient room temperature data versus room temperature set point data is used to automatically increase fan speeds to achieve a room temperature customer set point during the forced air systems “on” cycle and or the damper position can be changed to allow more supply duct air to be mixed with room air when the forced air system is “off”. An increased percentage of supply duct air versus room air can help the room achieve the homes ambient temperature when the forced air system is not operating. Fan speeds and damper positions can be changed, more or less, throughout the day to minimize fan noise, minimize filter usage and still meet consumer room air temperature set points.

In other embodiments, the ratio of supply duct air versus room air drawn into the air purification system is automatically adjusted to minimize the supply duct air percentage when both supply duct and room air are simultaneously drawn into and through the air purification system. By reducing supply duct air within the operational limits of the supply duct pressure thresholds and while also maintaining a minimum flow requirement from the supply duct to maintain positive room pressure for room ventilation, the filtering of the dirtiest air, coming typically from the supply duct, can be minimized to extend filter life. Additionally, when the air purification system temperature set point is set by the consumer to provide room air temperatures greater than or less than the ambient home temperature, it is advantageous that the air purifier system is capable of adjusting the damper position to use the smallest amount of supply duct air versus room air when both are drawn simultaneously to minimize temperature swings within the room during the forced air systems “off” cycle.

Furthermore, in some embodiments the supply duct pressure data collection when no filter is installed is compared to the data collected when a new filter is installed. Additionally, zip code information about regional utility rates are used to alert and report utility costs associated with the inherent air flow restriction many forced air system filters create. In some embodiments, costs associated with a dirty filter versus a clean filter are also presented to the consumer. As such the air purification system can “pay for itself” by saving energy and suggesting energy saving actions. The system’s prompts also elevate energy conservation awareness and actions. Even without the having the user follow a series of installation prompts to determine the room pressure threshold, the purifier system can log supply duct pressure data and extrapolate to recognize filter type changes as well as dirty filter replace-

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ments and prompt questions of the user to confirm changes and potentially cause the user to rethink some filter choices.

In these embodiments, when the ambient air pressure **144**, exceeds the maximum room pressure threshold, a visual or audio alert is provided to the consumer suggesting to open a door to the room. Opening a room door improves air circulation within the house. Opening a door can also provide room pressure relief. The room door being shut may not allow enough air to exit the room causing excessive ambient air pressure. Furthermore, the air may not be available to externally located intake vents for circulation and re-entry/return into the forced air system. Other ways to reduce room air ambient air pressure **144** include reducing the speed of the fan **114** or moving the bypass damper system to an air filtering mode which draws in less air from the supply duct inlet **104** and more from the room air inlet **102**, such as the position illustrated in FIG. 2A. In either instance (or a combination of both) the amount of supply duct air entering into the room is reduced thereby reducing the room air pressure. It is noted that enough open registers in other rooms will sometimes simply supply more air; under these circumstances reducing the possibility the forced air system is likely to experience a restricted return air flow.

When the pressure sensor measures supply duct air pressure **142**, it can take a variety of actions depending on the pressure threshold(s) reached. For instance, when a first pressure is exceeded, a lamp may turn on to alert the user. In some embodiments, the bypass damper system will move to the air filtering mode illustrated in FIG. 2B, wherein only air from the supply duct inlet **104** is received. In some instances this will allow for sufficient pressure relief (i.e., the first pressure threshold will no longer be exceeded.) In other embodiments additional action will be taken. For instance if a second pressure threshold is reached or if the first pressure is not relieved during after a period of time, e.g. 1 minute or 5 minutes for example, the speed of the fan **114** may be increased to relieve the excess supply duct pressure. It is noted that an increased fan **114** speed is used to relieve supply duct over-pressurization when the bypass damper system is in any of its air filtering positions, although the position shown in FIG. 2B is the most effective. In some embodiments, the lamp is left on or is set in a flashing mode when the fan speed increases to notify the user that a pressure fix is in progress. Then in the event that an increased fan **114** speed also does not relieve the supply duct pressure or a third threshold is reached, the pressure control mechanism **129** will activate the bypass damper system **110** to move from an air filtering mode position to a bypass mode position allowing the supply duct air to pass directly into the room without requiring (or sometimes by actively disallowing) the air to pass through the filter(s) **108**.

In some embodiments, an electronic controller **132**, such as a microprocessor based printed circuit board control is utilized to control the fan **114** speed in accordance with the pressure readings as described in the above paragraphs. For instance, in some embodiments, the electronic controller **132** is configured to automatically activate movement of the bypass damper system **110** from the at least one air filtering mode position to the bypass mode position when the measurement of the pressure sensor **142** air exceeds a bypass threshold.

In some embodiments, the controller **132** additionally or alternatively automatically monitors supply duct temperature with a temperature sensor **146a** at or near the supply duct inlet **104** area, measures ambient room temperature with another temperature sensor located outside the unit **146b** (or isolated inside the unit to only measure ambient air temperature), and controls the speed of the fan **114** or the placement of the

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bypass damper system in accordance with the temperature readings. The two temperature sensors **146** supply temperature data that is interpreted by the controller **132**. In some embodiments, the controller manages room temperature by adjusting the speed of the fan **114**. For instance, when the room air temperature sensor **146b** measures room air temperature to be above or below a target temperature for the room (as set by the user or determined automatically by the air purification system **100**) the speed of the fan **114** is increased to pull more temperature treated air from the forced air system. Then in the event that an increased fan **114** speed also does not allow the room to be within a predetermined range of the target temperature after a period of time (e.g. 30 seconds, 5 minutes, 10 minutes, or 20 minutes), the electronic controller **132** will activate the bypass damper system **110** to move from a first air filtering mode position (FIG. 2A) to a second air filtering mode (FIG. 2B). In some embodiments, when the temperature sensor **146b** measures room air temperature and a first temperature condition is met the bypass damper system is configured to be in a first air filtering mode position (FIG. 2A), and when a second temperature condition is met the bypass damper system is configured to be in the second air filtering mode position (FIG. 2B). When supply duct air is of sufficient temperature to add warmth or cool air to the room to achieve a target room temperature, the fan **114** speed and bypass damper system position is adjusted accordingly by a microprocessor program in the electronic control **132**. In some embodiments, the electronic controller **132** also activates user alerts and takes actions to provide safety and efficiency during the units operation and/or while it is installed.

In some embodiments, the temperature sensors **146** offer insight into the function of the forced air system derived from the temperature sensors ability to automatically measure supply duct air temperature by the first temperature sensor **146a** over time and log data including the coolest supply duct temperatures reached relative to the ambient room air temperature measured by the second temperature sensor **146b** in the cooling season and also the warmest supply duct temperatures versus the ambient room air temperature in the warm air required seasons. If the difference between the room air temperature and the supply duct temperature is not as great as recommended, then the central forced air system may need service. The air purification system **100** can automatically recognize the circumstances and report via alert or text message to the user, the need for service by means of the electronic controller **132**, which may display information on a display panel of the air purification system **100** or by wirelessly communicating information to a remote device such as a remote control, a personal digital assistant, a tablet computer, or desktop computer (e.g., in some embodiments, the alert is in the form of an email reminder). For instance, a heat pump often requires additional refrigerant after a few years of service. The user may not know this or forget to add the refrigerant. When the heat pump does not get serviced, electricity is unnecessarily wasted and possibly the forced air system components have been or may become damaged. Thus, the benefits of data automatically collected from the temperature sensors **146** extend beyond the air purification system itself. The data automatically collected from the temperature sensors **146** can thus be used to provide information relevant to the forced air system's overall refrigerant charge and or heating capability which, when addressed can result in energy savings for the user. Additionally, a forced air technician may choose the temperature differentials to be monitored during installation of the air purification system **100**. In this manner, older equipment can be tuned up and a field evaluation of the optimum performance versus service call criteria

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can be considered and set points can be adjusted accordingly by the user or the technician. In addition, the air purifier may automatically follow certain generalized forced air system guidelines and performance expectations and report abnormalities for customer consideration.

FIG. 2A is a side section view of the embodiment of FIG. 1, wherein the bypass damper system is in a first air filtering mode position. As illustrated in FIG. 2A, in the first air filtering position the first blade 124 is in a third position, which uncovers the room air inlet 102 allowing air to pass from the room air inlet 102 into the air purification system 100. The second blade 126 is in a fourth position, which seals against the first blade 124 covering the through holes (or slots) 128 in the first blade 124. In some embodiments, the second blade 126 is configured to fit within the exterior boundaries of the first blade 124. In some implementations the second blade 126 seals against the first blade 124 such that the two act as one damper. As illustrated in FIG. 2A, in a first air filtering mode, air inlet damper 122 acts as one damper and is positioned to allow supply duct air and room air to enter the air purification system 100 simultaneously. Specifically, this air filtering position of the air inlet damper 122 guides air from the supply duct inlet 104 and air from the room air inlet 102 to both pass through the filter 108 and the outlet, 106 and simultaneously impedes air from passing from the supply duct inlet 104 to the room air inlet 102. The impeding is done by the second blade 126 covering and thus not allowing air to pass through the holes 128 in the first blade 124. It is beneficial that the air passing from the supply duct inlet 104 is impeded from exiting out of the room air inlet 102 because, during an air filtering mode the dirty air from the forced air system is cleaned most effectively by being guided to be drawn into and pass through the filter 108 before it ever enters the room.

In some embodiments, the second blade 126 is held in place, sealed against the first blade 124 by means of an electro magnet 136. The electro magnet 136 is of sufficient force to hold the second blade 126 against the first blade 124 and to compress the spring 134. In some embodiments, tab 138 is placed in a position to force the second blade 126 into contact with the first blade 124 and the electro magnet 136 when the first blade 124 moves to the third position, as illustrated in FIG. 2A. When the second blade 126 contacts the first blade 124, the energized electromagnet 136 holds the second blade 126 against the first blade 124. In some embodiments, the tab 138 is adjustable to accommodate differing air filtering mode positions. For instance, in some implementations the tab 138 is spring loaded as illustrated herein.

FIG. 2B is a side section view of another embodiment of the air purification system 100, wherein the bypass damper system 110 is in a second air filtering mode position distinct from that shown in FIG. 2A. As illustrated in FIG. 2B, in a second of a plurality of air filtering positions the first blade 124 is in the first position, which covers the room air inlet 102 and the second blade 126 is in a fourth position, which seals against the first blade 124 covering the through holes (or slots) 128 in the first blade 124, as such the two blades act as one damper that does not allow air to pass from the room air inlet 102 into the air purification system 100. In other words, the bypass damper system includes a second air filtering mode position, wherein the second air filtering mode position guides air only from the supply duct inlet through the filter and the outlet. As illustrated in FIG. 2B, in this second air filtering mode, air is drawn exclusively from the supply duct inlet 104. Specifically, this air filtering position of the air inlet damper 122 guides air from the supply duct inlet 104 to pass through the filter 108 and be expelled from the outlet 106, and

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simultaneously blocks air from passing from the room air inlet 102 into the air purification system 100.

It is noted that when the first blade 124 and second blade 126 are in the second air filtering position, such that air is drawn exclusively from the supply duct inlet (e.g., 104a), this air may be drawn at greater than normal rates typically found in forced air systems and subsequently discharged into the room. As such, the room may be heated or cooled more quickly than if air were also being drawn from the room using the room air inlet 102. Furthermore, in some embodiments, a greater percentage of air is drawn into the room from the forced air system than would be the case under normal circumstances (e.g., under circumstances where no air purification system 100 is used). As such, in addition to quickly achieving a desired room temperature, another advantage of drawing a greater percentage of air from the force air system is to control a positively pressurized room. In a positively pressurized room, potentially contaminated air from outside the room is kept out, and eventually all or essentially all of the air in the room will have been filtered through the air purification system 100 creating a room full of clean air.

In some embodiments, the bypass damper system 110 is controlled by a motor 148, which controls movement of the first blade 124 the first position to the third position and various positions in-between (e.g., as illustrated with respect to FIGS. 1 and 2A). However, FIG. 2B illustrates a simpler embodiment in which a suitably light weight and mass balanced bypass damper system 110 that is motor-less. The movement of the first blade 124 is controlled by air flow pressure and/or a gravity bias. Specifically, the bypass damper system 110 is gravity biased to lie against the room air inlet 102, for instance in some embodiments the gravity bias is 51/49 in favor of lying against the room air inlet 102. In some embodiments, the center of mass of the first blade 124 is controlled by placement of the electro magnet 136 to bias toward the room air inlet 102. Similarly, in some embodiments, the center of mass of the first blade 124 is controlled by weight on an extended lever placed toward the supply air inlet to make the current damper fulcrum point have at least a slight gravity bias that rests at room air inlet 102 when the unit is off.

The motor-less gravity biased bypass damper system will allow air to enter from the room air inlet 102 when the fan 114 pulls as much or more air than is supplied from the supply duct inlet 104. However, with the air from the supply duct flowing behind the air inlet damper 122, and in some embodiments, when combined with its gravity bias toward the room air inlet 102, the room air inlet 102 will close as soon as the air flow from the supply duct is greater than the current speed of the fan 114. In some embodiments, the motor-less gravity biased bypass damper system is then manual reset by the user by pushing the air inlet damper 122 away from the room air inlet 102. In other embodiments, when the pressure from the air from the supply duct 104 decreases, the air inlet damper 122 will naturally tilt back allowing air to enter from the room air inlet 102 again, and thus the air purification system will again receive air from both the supply duct inlet 104 and the room air inlet 102.

As such, when the supply duct pressure is in normal ranges with this unit turned off (or when the unit is on with a fan speed incapable of taking all the supply duct air being pushed by the forced air system blower) the 2 piece air inlet damper 122, bound together by the electro magnet 136, is forced against the room air inlet 102, which seals and prevents air from the supply duct 104 from escaping into the room through the room air inlet 102. Thus only the supply duct air flows through the filters 108, and exits through the outlet 106.



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However, if the supply duct pressure reaches a bypass threshold an automatic disengagement occurs: the magnet releases **136** the second blade **126** such that supply duct air can exit through the room air inlet **102** into the room through the holes **128** in the first blade **124** eliminating excess pressure.

In some embodiments, the automatic disengagement occurs as follows: the second blade **126** is held in place, sealed against the first blade **124** by means of an electro magnet **136**. In some embodiments, this electro magnet **136** is wired to a pressure sensor **130** (such as a MPL 9371 “snap action” switch described below). The pressure sensor **130** has a bypass pressure threshold, and exceeding the bypass threshold opens switch contacts thereby cutting the electricity to the electromagnet **136**, which in turn releases second blade **126** from the first blade **124** thus putting the bypass damper system **110** into the bypass mode position, as illustrated in FIG. 1, which allows air from the supply duct **104** to escape through the room air inlet **102** and relieve pressure.

FIG. 2C provides a perspective view of the bypass damper system **110** of the air purification system **100** to more clearly illustrate the functionality of some of the components. This figure illustrates the bypass mode position where the first blade **124** and the second blade **126** are separated from each other. Air entering the system from the supply duct inlet **104** is guided directly into the room, without requiring passage through the filter and outlet, by the second blade and through the through holes (or slots) **128** in the first blade **124**.

FIG. 3 is a side section view of an air purification system in accordance with another embodiment of the present invention, wherein the bypass damper system is in a bypass mode position. The embodiment of FIG. 3 is simpler than that of FIGS. 1, 2A, and even the 2B in that the bypass damper system relieves pressurization in the supply duct without using a two blade air inlet damper **122** (see FIG. 1).

FIG. 3 illustrates a bypass damper system **110** that includes a bypass damper **302** (see FIG. 5) and an air inlet damper **304**. A protective grill **308** covers the bypass outlet **306** to protect the internal mechanisms of the air purification system **100**. The air inlet damper **304** is similar to the air inlet damper **122** of FIG. 1 except that it is a single piece damper without through holes or slots. FIG. 3 illustrates this bypass damper system **110** in the bypass mode position, which allows air to pass into the room through one or more bypass air outlets **306** distinct from the room air inlet **102**. The bypass air outlet **306** is configured to be covered and uncovered by the bypass damper **302** for excessive supply duct air pressure relief.

In the bypass mode position, the bypass damper **302** is positioned to expose the one or more bypass air outlets **306**. Furthermore, the air inlet damper **304** is positioned to seal the room air inlet **102** and stop air from passing from the room air inlet **102** into the air purification system.

FIG. 3 also illustrates a check damper **303** shown in a closed position. The check damper **303** is a light weight damper that is gravity biased to remain in the closed position when the bypass damper system **110** is in the bypass mode position. The check damper **303** uniquely placed to prevent flowing supply duct air from sweeping contaminants off the face of the filter **108** and then further carrying them into the room (e.g. through the bypass air outlet **306**). As illustrated in FIG. 4, the check damper **303** opens when the bypass damper system is in an air filtering mode position. Essentially, the bypass damper is light enough that it “flies” and thus opens when sufficient air is provided along the air distribution panels (e.g., whenever the air purification system **100** is in an air filtering mode position.) The check damper **303** is installed to the inside of the air distribution channel to disrupt a complete path of air from mixing with and stripping contaminants off of

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the face of the filter and potentially entering the room. It is noted that OSHA required forced air technicians to wear a face mask and disposable gloves when handling dirty filters because the face of the filter is a bio nest of contaminants. As such, the check damper **303** is included to ensure that the air purification system does not send concentrated contaminants into the room on a stream of air. This is a useful safety feature to be included in this and any unit described herein that connects to a supply duct of a forced air system. For instance this ensures that concentrated contaminants do not enter the room even when the forced air system is operating and the room air purifier is turned off or unplugged.

FIG. 4 is a side section view of an air purification system **100** in accordance with the embodiment of FIG. 3 the present invention, wherein the bypass damper system **110** is in an air filtering mode position. In the air filtering position, the bypass damper **302** is positioned to seal the one or more bypass air outlets **306**. FIG. 4 represents a closed position for the bypass damper **302** by the partial shading. In the air filtering position, the air inlet damper **304** is positioned to expose the room air inlet **102** and guide air from the supply duct inlet **104** and air from the room air inlet **102** to both pass through the filter **108** and the outlet **106**, and impede air from passing from the supply duct inlet **104** to the room air inlet **102**.

FIG. 5 is a rear section view of the embodiment of FIG. 3, wherein the bypass damper system **110** is in an air filtering mode position. In FIG. 5, the bypass damper **302** is shown closed, sealed against a bulk head covering the bypass air outlet **306** and held in place by Electromagnet **310**. A spring **312**, is biased to open the bypass damper **302** fully when the electromagnet **310** is de-energized (such as when a bypass threshold is exceeded). The re-set lever **314** is configured to push against the bypass damper **302**, urging it in position to be in contact with, or near enough contact for the energized electromagnet **310** to connect to and hold the bypass damper **302** in place covering the bypass air outlet **306** and overcoming the force of the spring **312**. In some embodiments, the re-set lever **314** is in, more or less, continuous contact with the bypass damper **302** and as the lever **314** rotates through positions which place the air inlet damper **304** in its positions the bypass damper **302** is automatically re-set. This automatic re-set allows the bypass damper **302** to open inwardly and allows the protective grill **308** (see FIG. 3) to be attractive and placed on the outside of the unit.

FIG. 6 is a rear section view of the embodiment of FIG. 3, wherein the bypass damper system **110** is in a bypass mode position. In the bypass mode position, the bypass damper **302** is in an open position (uncovering the bypass air outlet **306**). The re-set lever **314** is configured to rotate as the air inlet damper **304** is positioned to reduce air flow from the supply duct inlet **104** (See FIG. 3), and the re-set lever **314** stays in contact with the bypass damper **302** until the bypass mode position is attained. FIG. 6 shows the added optional feature of a re-set lever **314** being equipped with a lever spring **316**, which is designed to stay in full contact with the bypass damper **302** in the event alignment damage or other tolerances are exceeded for various reasons. Thus, the re-set lever **314** will automatically adjust to minor dimensional changes by use of the lever spring **316** and therefore ensure that the electromagnet **310** is adequately positioned to re-engage the bypass damper **302** when energized. Furthermore, in FIG. 6, a low friction (e.g., nylon) tip **318** is also shown at the end of the re-set lever **314**. Friction reduction is beneficial especially during the initial arching/rotation movement of the reset lever **314**. A 90 degree tab **320** on the bypass damper **302** is also shown and runs a length of the bypass damper **302**, whereby the spring loaded reset lever **314** has the tab **320** to prevent



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accidental disassembly of the reset lever **314**. In some embodiments, the lever spring **316** may also have an internal stop to limit the overall length of the reset lever **314** and eliminate accidental disassembly.

This bypass damper system **110** of FIGS. 3-6, which includes a bypass damper **302** (see FIG. 5) and an air inlet damper **304**, is an alternate version of the bypass damper system **110** illustrated in FIGS. 1-2B. However, both types of bypass damper systems **110** are configured to relieve excess supply duct pressure in the bypass air outlet bypass mode position. For instance, as illustrated in FIG. 6, when the pressure from the air in the supply duct inlet **104** exceeds a bypass threshold the bypass damper **302** moves to its spring open position allowing air from the supply duct to pass directly into the room.

FIG. 7 is a rear section view of an air purification system **100** in accordance with another embodiment, wherein the bypass damper system **110** is in a bypass mode position. FIG. 7 is quite similar to the embodiments illustrated in FIGS. 3-6 except that there are two bypass dampers **302a** and **302b**. This two bypass air outlets **306a** and **306b** allow for quicker and more comprehensive excess supply duct pressure relief.

FIG. 8 is a rear section view of an air purification system **100** in accordance with yet another embodiment, wherein the bypass damper system **110** is in a bypass mode position. FIG. 8 is similar to FIG. 7, in terms of having two bypass dampers **302a**, **302b** and two bypass air outlets **306a**, **306b**. However, FIG. 8 illustrates an embodiment in which the bypass dampers **302a**, **302b** open outward from the air purification system **100** rather than internally. Each is still held by an energized electromagnet **310a**, **310b** until release conditions are encountered and executed (e.g., the bypass threshold is met). One advantage of the FIG. 8 embodiment is that the bypass dampers **306a**, **306b** are configured to be manually operated by a user pushing them closed until they latch, either by hand or by broom stick depending on whether the air purification system is covering a floor, wall, or ceiling supply duct register/vent.

FIG. 9 is a side section view of a gravity bypass damper system **900** separate from the room air purifier unit. The gravity bypass damper system **900** is installed into the supply air side of a forced air zone, and is configured to relieve any excess pressure that may occur in the supply duct of the forced air system. The gravity bypass damper **900** illustrated in FIG. 9, includes a damper blade **902**, a communication module **904**, adjustable weights **906**, and a micro-switch **908**. In some embodiments, the micro switch **908** is configured to turn on the communication module **904** when triggered by movement of the damper blade **906**. The adjustable weights **906** are used to control the damper blade **902** into the bypass, by limiting how much pressure is allowed to build up from the forced air supply **910** before the damper blade **902** moves from a first position (shown in solid lines) to a second position (shown in dashed line) where the supply duct air is routed out of the forced air system (such as into the basement).

Re-routing forced air system supply air **910** is one purpose of the gravity bypass damper system **900**, but this system also allows for other more elegant solutions explained below. First, it is illustrative to understand when the bypass damper system **900** is utilized. As explained above, some room air purifiers (e.g. System **100**, FIG. 1) are equipped with room temperature controls, and are capable of producing independent room zones. The room air purifiers are configured to work in concert with the independent operation of the forced air system. As such, temperature settings applied by a user at each room air purifier unit could be set to conflict with the home's overall thermostat control. Furthermore, although the

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room air purifiers are configured to typically, continuously, draw approximately one half of what a normal supply duct would normally supply, situations could arise where multiple room air purifiers working in combination will control the supply ducts in such a way to create a situation where supply duct pressure also exceeds a set point. For instance, when the room air purifier units are used to control temperatures significantly above or below the temperature set point of the whole house thermostat, an over pressurization of the forced air supply may occur. For example, if the house thermostat is set at 72 degrees Fahrenheit in the summer, the upstairs zone has 3 bedrooms, and all three bedrooms have room air purifier units installed and set to 75 degrees Fahrenheit, after a time, the bedrooms will warm up to 75 degrees Fahrenheit and then not take much or any 72 degree Fahrenheit additional supply duct air in order to keep the room temperatures at or near the set point of 75 degrees Fahrenheit. The limiting of the supply duct air in all three bedrooms could cause over pressurization in supply duct. Similarly, sometimes users may shut doors and try to create extreme temperature zones room by room, which also may create over pressurization in the supply duct.

As described with respect to the previous figures, one solution is that when a bypass pressure threshold is reached, the air purifier units (e.g. System **100**, FIG. 1) will move from an air filtering position to a bypass mode position. Alternatively another solution is that in some embodiments, prior to a bypass pressure threshold to be reached, the gravity bypass damper **900**, as shown in FIG. 9, will dump the conditioned air, outside, into the basement, or into a return air duct. In some embodiments, the gravity bypass damper **900** is configured to relieve the excess pressure by moving a damper blade **902** from a first position to a second position where the supply duct air is routed out of the forced air system (such as into the basement). The threshold for this movement of the damper blade **902** is controlled by the adjustable weights **906**. However, in some embodiments, the gravity bypass damper **900** of FIG. 9 also offers another more elegant solution to supply duct over pressurization and temperature set point mismatches than dumping the air into the basement (or similar external location) as explained below.

FIG. 9 illustrates a bypass damper **900** that utilizes a communications module **904**. In homes with multiple zones, a by-pass damper **900** may be installed in the supply side of the forced air system before the forced air reaches any rooms. In some embodiments, this bypass damper **900** is a gravity type/weighted, is mechanically based or has pressure sensors to activate a motor drive to open it to a specific point, and then later, closed again at another point. This by-pass damper **900** relieves excess duct pressure to insure a proper amount of air is moved by the forced air system and re-circulated through the system. The by-pass damper **900** may dump air into a basement to be picked up by a return air duct, for example or the by-pass damper can dump the air directly into the return air ducting at a preferable location in the return duct. The by-pass damper **900** is one method to insure proper movement of air through the forced air system while also not exceeding the forced air systems duct pressure manufacturer's specifications. Sometimes a by-pass damper **900** is not enough and forced air systems can be abused to the point of failure. Dirty filters, for example, can restrict the amount of air circulation and cause cooling components to freeze and heating components to overheat and possibly become damaged.

The forced air system professional can install forced air systems with alarms and service reminders, like filter monitors and cooling coil and heat exchanger temperature sensors to warn or even shut down the system, if necessary, to prevent

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damage to the system. These sensors can be installed as accessories at any time, as well. Supply duct pressure on the other hand may not even be checked when a system is first installed.

FIG. 9 illustrates a bypass damper **900** that utilizes a communications module **904** to overcome some of the above mentioned issues. The communications module **904** capable of communication with room air purifier units in the home by utilizing the communication module **904**. For example, in some embodiments, the communication module **904** is in communication with an electronic controller **132** of one or more room air purifier units (e.g. FIG. 1, **100**). Rather than waste conditioned air by dumping it into the basement if supply duct pressures are exceeded, the communication module **904** communicates with the room air purifier unit's electronic controller **132** to change its air flow profile.

In some embodiments the bypass damper **900** detects the by-pass damper movement **902**, (e.g. by means of the consequent movement of the micro switch **908** from an off position shown in solid lines to an on position shown in dashed lines). The communication module **904** then begins communicating with the air purifier unit's electronic controller **132** to change its air flow profile and thus allow the bypass damper **902** to return to its original position (of allowing forced air system supply air **910** to reach the room air purifiers).

For instance, in some embodiments, an individual room air purifier unit in communication with the communications module of the **904** of the bypass damper **900** will respond by:

Increasing fan speed. In some embodiments, the fan speed is increased to double or triple a single supply duct's air flow to mimic multiple supply ducts in full flow operation.

Changing bypass damper system **110** position to draw more supply duct air.

Or both.

In some embodiments, if a bypass damper system **110** of one or more of the units **100** has already moved into a bypass mode position, the bypass damper's **900** communication module **904** directs the unit's damper system **110** to reduce or eliminate the bypass mode air flow and draw more supply air through the unit **100**.

As such, in some embodiments, the room air purifier unit's bypass damper system **110** working in conjunction with an increased fan **114**, eliminates or minimizes the discharge by the bypass damper **900**. As such, pressure is relieved in the forced air system supply air without wasting it by dumping it into the basement or elsewhere outside the system.

Furthermore, in some embodiments multiple room air purifier units can coordinate to perform these functions. As such, there is achieved little or no undesirable impact on room temperature versus user controlled room temperature set points thus overcoming the issues described previously. In some embodiments, the communication module **904** individually directs the activities of multiple air purifier units to achieve an overall optimal outcome for the home.

FIGS. 10A-10F illustrate the components of the air purification system **100** that provide convenient filter disposal. In some embodiments, the system **100** includes a filter door **1002** (FIG. 1 OA) located at the top of the unit for ease of filter changes. The door **1002** exposes a filter compartment **1004**. The door handle is also configured to hold a disposable bag **1006** and a tool **1008** to remove the filters **1010** (FIG. 10B). The tool **1008** removes the filters with a lift handle **1012** and a foot **1014** for the filters to sit on. In some embodiments, the tool **1008** is of sufficient thickness (e.g., 0.0625 in) to put a squeeze on the filter(s) **1010**, when inserted. The squeezing action creates a seal around the filters **1010** allowing for easier

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disposal. Furthermore, in some embodiments the tool **1008** has permanent seals on either or both sides to create the filter seals needed (FIG. 10D).

It is noted that the top access filter door **1002** also helps remove filters and insert them into the bag **1006** while minimizing contaminants dropping off of the filter **1010** and into the room. A bag catch may also serve as a door opener handle and this latch can hold the bag **1006**, which is especially helpful in a ceiling mounted air purifier servicing a ceiling register. Consequently, in action, the filters **1010** slide out of the air purifier and into the bag **1006**. The filter removal tool **1008** does triple duty by lifting out individual filters **1010**, holding the bag with a lift handle **1012** and the tool **1008** also serves as a wedge between filters **1010** to enhance the seals between filters and against the filter housing.

FIG. 11 is a rendering of the top of an air purification system **100** in accordance with some embodiments. The top located outlet **106** is illustrated. The system may also include a communication panel **1102**. In some embodiments, the communication panel includes a display screen **1104**, which provides various communications/user messages **1106** to the user such as the current temperature of the room, the pressure in the system, and warning messages such as when a door should be opened or a filter should be changed. The communications panel **1102** may also include one or more input means such as buttons **1108**. The input means can be used to input set points for the system **100**, such as a desired room temperature. The system's microprocessor control **1110** is shown ghosted and includes a longer range communications antenna connection port plug **1112**. In some embodiments, the communications panel **1102** is provided on a separate remote control unit rather than on the top of the system **100**. This remote control unit is especially useful for ceiling mounted systems **100**. In such embodiments, the remote will communicate to the system **100** via an antenna **1114** attached to the connection port **1112**. In some embodiments, the remote may also communicate by wire. In some embodiments, the remote includes the room temperature sensor to provide means to optimally locate the sensor to reflect conditions similar to the homes forced air system thermostat, e.g. a median distance above the floor.

FIG. 12 is a perspective view of an air purification system **100** in accordance with still another embodiment. The bypass damper system **110** is a gravity bypass damper **322**. With the gravity bypass damper **322** no pressure sensor is needed to activate the bypass damper because it is held in place by gravity and a certain amount of weight **1202**—corresponding to the bypass threshold releases the damper **322**. In some embodiments, a latch **1206** is released and a spring **1204** opens the gravity bypass damper door **1208** when the threshold is reached. Then the door **1208** is manually closed and the latch **1210** is reattached by the user. In other embodiments, the damper door **1208** flies freely, and it automatically closes by gravity when the air pressure is relieved. This embodiment is simpler and cheaper to implement than some of the aforementioned bypass damper systems **110**. The gravity bypass damper **322** is hinged to work with gravity for vertical, horizontal or even upside down mounting requirements. It is noted, that other than the simpler gravity bypass damper **322**, in some embodiments the elements of the air purification system **100** in FIG. 12 are the same as those illustrated in FIG. 3. For instance, the unit includes the air inlet damper **304** and the check damper.

FIG. 13 illustrates an enhanced air distribution channel **1300** for the air purification system. The air distribution channel **1300** illustrated in FIG. 13 is similar to the air distribution panel **116** illustrated in FIG. 1 except that it has several

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improvements. For instance the air distribution channel **1300** includes a “V” shaped louver **1302** attached to a hinge **1304**. The “V” shaped louver **1302** is configured so that when the air flow on one side is of the distribution channel plate **1306** is greater than the other side, the point of the “V” of the louver **1302** will move into the flow of air on the higher flowing side. This movement will consequently restrict the flow of air to that side and therefore approximately equally split the airflow on either side of the channel plate **1306**. In some embodiments, louver **1302** is configured to optimally operate between 100 and 250 cfm. Furthermore, the top of the air distribution channel plate **1306** has a rounded edge **1308** to reduce air velocity noise and enhance flow over the top and into the filter area. Finally, as illustrated more clearly in FIG. **15**, in some embodiments, the air distribution channel **1300** also includes laminar flow channels **1310**.

FIG. **14** is similar to FIG. **13**, but also includes the addition of a check damper **303**. The addition of the check damper **303** influences the flow of air along the distribution channel plate **1306**, as all of the air first passes on one side and is then distributed across the filters **108** on the opposite side of the distribution channel plate **1306**. It is noted that any of the models illustrated and described above (e.g., the models illustrated in FIG. **1**, FIG. **3**, FIG. **7**, FIG. **8**, and FIG. **12**) can use the FIGS. **13** and **14** air distribution channel **1300** with the laminar flow channels **1310**, and the “V” shaped louver **1302** and the rounded edged **1308** together or independently. The check damper configuration is advantageous in all of these models to block the release of contaminants from the face of the filters **108**. The check damper **303** is especially advantageous in any model that does not, by other means, block supply duct air from mixing with and releasing contaminants on the face of the filters **108** into the room during certain circumstances or malfunctions. For instance, it is especially advantageous in the FIG. **3** and FIG. **12** models for example.

FIG. **15** is a perspective view of the air purification system's front panel **1500** with the air distribution channels **1300** attached to it. This figure illustrates laminar flow air channels **1310** on each side of the air distribution channel plate **1306**. As discussed above, the laminar flow air channels **1310** reduce air turbulence and provide a smoother flow of air on both sides of the air distribution channel plate **1306**. As illustrated in FIG. **15**, the total air flow is divided among the multiple laminar flow air channels **1310** and thus delivered equally across the face of the filter. In some embodiments, the laminar flow air channels **1310** are not evenly spaced as shown herein, but are still spaced so as to distribute air approximately evenly across the face of the filter. As such the laminar flow air channels **1310** are configured to use the filters full surface area most effectively for collecting, absorbing, and providing optimum air flow over the broadest possible surface area of the filter. For instance, without the laminar flow air channels **1310**, the majority of the air may pass through the bottom third of the filter causing accelerated flow rates and less effective contaminate collection and absorption.

FIG. **16** is an illustration of an air purification system pressure control mechanism in which an electro magnet **1602** holds an actuator bar **1604**, which is biased with a spring **1606** in an air filtering position (see right pointing arrows). The actuator bar **1604** moves freely inside nylon bushings **1608**.

In some embodiments, when the system **100** has power and is turned “on” all of the louvers **1610** over the air inlet **102** are shut. Then if power is cut, if the unit is turned “off”, if a supply duct over-pressure sensor exceeds a high limit set point, or some other particular failure mode takes place, the electro magnet **1602** can release the spring **1606** loaded which

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releases actuator bar **1604** and all of the adjustable slide louvers **1610** are pushed to a fully “open” position simultaneously (see left pointing arrows.)

FIG. **17** is an illustration of the air purification system with supply duct outlets that have louvers **1702** with exterior handles. As such, a user can manually reset the actuator bar against the electromagnet (in FIG. **16**) by fully sliding any of the louvers **1610** to a fully closed position when the electromagnet is energized to hold the actuator bar in a normal ready state.

FIG. **18** illustrates another embodiment of an air purification system **100** in which a second damper system **1802** is utilized in addition to the bypass damper system **110** described with respect to FIGS. **1** and **2A-2C**. The second damper system **1802** allows air to be supplied from a forced air system supply duct inlet **104a** and an additional supply air inlet **1804**. The blade **1808** is configured to cover and uncover the forced air system supply duct inlet **104a** and is configured to cover and uncover the additional supply air inlet **1804**. The second damper system can be moved from obtaining all air from the forced air system supply duct inlet **104a**, all air from the additional supply air inlet **1804** or various combinations of both.

In some embodiments, the second supply air inlet **1804** is configured to receive air from outdoors/outside the home or from another room. It may be beneficial to draw air from outdoors because the outside air quality measured by a sensor, timed or manually triggered may be better than the indoor air quality. For instance, the fresh outside air may not have some of the household chemicals like cleaning aerosols or fresheners sometimes found in high concentrations indoors. Therefore, in some instances, outside air is higher quality air than room or supply duct air and would require less filtering. As such, in some embodiments, it is beneficial to allow a system **100** to draw outside air for the purpose of providing higher quality air into the room or room air purification system.

Furthermore, in some embodiments, the air outside the room or the outdoor ambient air may be useful to help achieve a desired room set point temperature. For instance, if room air set point temperature is 72 degrees Fahrenheit and the room is currently at 78 degrees Fahrenheit, if outside/outdoors air is less than 78 degrees, for example 65 degrees Fahrenheit, then outside air can be utilized to reach the set point and minimize the necessity for air conditioning. Thus, in some embodiments, utilizing the outside air is a cleaner and greener/more energy efficient way to achieve a desired room temperature and air quality level.

For example, the second damper system **1802** may be used as follows: when the outdoor air from the supply air inlet **1804**, as measured by the first temperature sensor **1812a**, is cooler than the air from the supply duct inlet **104a**, as measured by a second temperature sensor **1812b**, and is also cooler than room's ambient temperature, as measured by the third temperature sensor **1812c**, (and less than the user's desired set point temperature for the room), then the second damper system **1802** would move the blade **1808** to close off the supply duct inlet **104a** completely and simultaneously move the blade **1808** to open the supply air inlet **1804** such that the cooler air from the outside would help cool a room without utilizing the air conditioner and thus saving energy. Similarly, warm outside air can be used to heat a room to a desired temperature.

In some embodiments, the outside air, filtered and released into the room would provide continuous positive pressure to the room while the supply duct inlet **104a** is closed.

In some embodiments, the temperature sensors **1812a** and **1812b** are positioned inside the system **100** and thus may

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require that blade **1808** of the second damper system **1802** periodically unseal the respective inlets in order to sample the temperature of the air in the respective supply ducts. In other embodiments a small hole is provided in each respective damper, this controlled air leakage through the holes is used by the temperature sensors **1812a** and **1812b**.

In some embodiments, other outside air sensors are used, which include humidity sensors and air pollution sensors. In some embodiments, when humidity sensors are used, and a significant disparity in humidity is measured between the outside air and the supply duct air, air from the 2 sources can be blended to achieve a desired temperature and humidity level. For instance, the second damper system **1802** may use cool humid air from the outside and blend it with drier indoor supply duct air in a ratio to not overly humidify the room or the house for that matter while still achieving a desired temperature for the room.

FIGS. **19A-19B** are a flowcharts illustrating a methods **1900** of providing pressure relief and alerts in accordance with some embodiments. The description of these methods is meant to provide an easy reference for the method of operation of the air purification systems described with reference to FIGS. **1-18** above and is not meant to be limiting to only the methods described herein. These methods **1900** are typically governed by instructions that are stored in a computer readable storage medium and that are executed by one or more processors within the electronic controller **132**. Each of the operations shown in FIGS. **19A-19B** typically corresponds to instructions stored in a computer memory or non-transitory computer readable storage medium. The computer readable storage medium typically includes a magnetic or optical disk storage device, solid state storage devices such as Flash memory, or other non-volatile memory device or devices. The computer readable instructions stored on the computer readable storage medium are in source code, assembly language code, object code, or other instruction format that is interpreted by the one or more processors of the controller **132**.

The method **1900** begins by providing an air purification system **1902**. As described above, the air purification system is configured to fluidly receive air from a forced air system, to fluidly receive air from a room, and to expel filtered air into the room. In some embodiments, the air purification system is initially installed to be in fluid connection with a forced air system in this step.

In some embodiments, one or more initial condition temperature measurements and/or pressure measurements are taken **1904**. In some embodiments, one or more initial air pressures are measured from the forced air system supply duct. Likewise, one or more initial room air pressures are measured. Furthermore, in some embodiments, one or more room air temperatures and supply duct temperatures are also measured.

Sometimes supply duct temperatures and/or pressures are measured at installation **1906**. Furthermore, sometimes and ambient room temperatures and/or pressures are also measured **1906**. In some embodiments, installation measurements are taken at various combinations of fan flow rates and damper positions **1908**. Likewise, in some embodiments, installation measurements are taken with room door open and closed **1910**. Furthermore, in some embodiment installation measurements are taken with and without a filter **1912**. In some implementations, the filter is new, while in other implementations the filter is used. For instance, in some embodiments, during the room air purification system installation with the customer follows prompts to open and close the room door(s), removing filters, installing new filters, etc. while the air purification system unit runs combinations of fan flow

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rates and damper positions, logs and interprets the data. The initial measurements are typically performed for only a diagnostic period of time, long enough to get a steady reading, e.g. 5 minutes.

Then the system determines a room profile which includes one or more thresholds in which actions will be performed **1914**. For instance in some thresholds determined include alert thresholds, thresholds which trigger an increased fan speed, damper filtering position thresholds (e.g. to move from one filtering position to another), and bypass thresholds. For instance, the initial temperature and/or pressure measurements are used to establish the maximum room pressure threshold value in some embodiments. It is further noted that in some embodiments flow measurements are adjusted to account for the air density effects caused by temperature and altitude. For instance, the initial pressure data collected is normalized to a range of standard temperatures in the supply duct.

Then after installation supply duct temperatures and/or pressures are measured **1916**. Furthermore, the ambient room temperatures and/or pressures are also measured **1916**. All of the measurements, both before and after the installation, are logged **1918**.

In some embodiments, various alerts are provided to the user based on the initial/installation measurements and the later measurements **1920**. The alerts are provided based on various thresholds being met. For instance, alerts may include: an air filter changing recommendation, a temperature setting recommendation, a repair/service recommendation, a room door opening recommendation, and a refrigerant replacement/addition recommendation. Furthermore, room air temperature can be compared to supply duct air temperature. For example, if the room temperature is 72 degrees Fahrenheit and the supply duct temperature cooling that room is only 8 degrees less (62 to 64 degrees Fahrenheit), then a "call for service" alert may appear. For instance, when there is only a 8 to 10 degree difference there may be an under-insulated supply duct or unsealed supply duct that goes through a hot attic or other reasons for this low delta in temperatures. After various measurements are run by the air purification system, the alerts may suggest that the consumer check the forced air system for malfunction (e.g. faulty coil or compressor). As such, the air purification system provides helpful, money and energy saving alerts to the user regarding not only the air purification system itself, but also alerts regarding the forced air system to which it is installed.

Furthermore, in some embodiments, a fan speed is automatically changed based on measured temperature(s) and/or pressure(s) **1922**. For instance, in some embodiments, the fan is automatically activated at high speed, while maintaining the bypass damper system in the at least one air filtering mode position when the air received from the supply duct inlet exceeds a minimum threshold **1924**. In other embodiments, the fan speed is automatically activated to a lower speed when the room door remains shut thereby forcing other room supply duct registers to increase flow and potentially return more air to the forced air system. For instance, when the supply duct temperature and the room air temperature difference reach a minimum difference between the two, e.g. 10 degrees Fahrenheit, over a shortened period of time, e.g. hours or maybe a few days, a coil may be freezing. This may be the result of a rapid loss of ac refrigerant or the room door being closed and the return air flow to the forced air system being just restricted enough, that in combination with other circumstances, causes a problem that appears relatively rapidly. Lowering the fan speed and or closing/restricting the air from the supply duct will force other supply duct registers in other

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rooms to get additional air flow from the HVAC system, return it, and thus help the situation.

In some embodiments, when the user's chosen room temperature is not being reached as intended, after several rounds of forced air cool or warm cycles, then the fan speeds are automatically increased, and the damper is automatically repositioned to draw more supply duct air in ratio to the room air when both are being drawn. This allows the room to equalize temperature with the rest of the home and more likely to get to the room set point desired temperature during the next forced air "on" cycle.

In some embodiments, the system automatically changes from a first air filtering mode to a second air filtering mode based on measured temperature(s) and/or pressure(s) **1926**. For instance, the system automatically expels filtered air received from both the room and from the forced air system when the air received from the forced air system does not exceed a bypass threshold and a first temperature condition is met **1928**. And the system automatically expels filtered air received from only the forced air system into the room when the air received from the forced air system does not exceed a bypass threshold and a second temperature condition is met **1930**.

However, when the bypass threshold is met, the system automatically releases unfiltered air received from the forced air system into the room **1932**. Thus, the release of pressure when a bypass threshold is met allows the air purification system to protect the forced air system from over pressurization.

Finally, in some embodiments, the air purification system also automatically resets itself back to an air filtering mode position when the bypass threshold is no longer exceeded. Thus, the system is configured to easily and safely provide cleaned, filtered, and purified air to a room quickly and efficiently.

The foregoing description, for purpose of explanation, has been described with reference to specific embodiments. However, the illustrative discussions above are not intended to be exhaustive or to limit the invention to the precise forms disclosed. Many modifications and variations are possible in view of the above teachings. The embodiments were chosen and described in order to best explain the principles of the invention and its practical applications, to thereby enable others skilled in the art to best utilize the invention and various embodiments with various modifications as are suited to the particular use contemplated.

What is claimed is:

1. An air purification system with over pressurization relief capabilities comprising:

a room air inlet configured to fluidly receive air from a room;

a supply duct inlet configured to fluidly receive air from a forced air system;

an air filter configured to filter air received from the room air inlet and/or the supply duct inlet;

an outlet configured to expel filtered air into the room;

a bypass damper system having at least one air filtering mode position and a bypass mode position; and

a pressure control mechanism configured to automatically move the bypass damper system from the at least one air filtering mode position to the bypass mode position when the air received from the supply duct inlet exceeds a bypass pressure threshold;

wherein the at least one air filtering mode position guides air from the supply duct inlet and air from the room air inlet to both pass through the filter and the

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outlet, and impedes air from passing from the supply duct inlet to the room air inlet; and

wherein the bypass mode position guides air from the supply duct inlet into the room without passage through the filter and the outlet.

2. The air purification system of claim 1, further comprising:

a pressure sensor configured to automatically measure air pressure received from the supply duct inlet; and

wherein the pressure control mechanism is configured to automatically move the bypass damper from at least one air filtering mode position to a bypass mode position when the pressure sensor's measurement air exceeds a bypass pressure threshold.

3. The air purification system of claim 2, further comprising:

wherein the pressure control mechanism includes an electronic controller configured to automatically activate movement of the bypass damper system from the at least one air filtering mode position to the bypass mode position when the pressure sensor's measurement air exceeds a bypass pressure threshold.

4. The air purification system of claim 3, configured to automatically move to bypass mode position when the air purification system is off.

5. The air purification system of claim 3, further comprising:

a fan having at least a high speed and a low speed; and wherein the electronic controller is configured to automatically activate the fan at high speed, while maintaining the bypass damper system in the at least one air filtering mode position when the air received from the supply duct inlet exceeds a minimum pressure threshold.

6. The air purification system of claim 1, wherein the pressure control mechanism is configured to automatically reset the bypass damper system by automatically moving from the bypass mode position to the at least one air filtering position when the bypass pressure threshold is no longer exceeded.

7. The air purification system of claim 1, wherein the bypass damper system includes a second air filtering mode position, wherein the second air filtering mode position guides air only from the supply duct inlet through the filter and the outlet.

8. The air purification system of claim 1, wherein the bypass mode position allows air to pass into the room through the room air inlet.

9. The air purification system of claim 8, wherein the a bypass damper system includes an air inlet damper comprising a two blade damper mechanism with a first blade and a second blade:

the first blade defining one or more through holes; and the second blade configured to cover and uncover the one or more through holes in the first blade;

wherein when the bypass damper is in the at least one air filtering mode position, the second blade covers the one or more through holes in the first blade; and

wherein when the bypass damper is in the bypass mode position, the second blade uncovers the one or more through holes in the first blade.

10. The air purification system of claim 1, wherein the bypass mode position allows air to pass into the room through one or more bypass air outlets distinct from the room air inlet.

11. The air purification system of claim 10, wherein the a bypass damper system includes a bypass damper and an air inlet damper:

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wherein in the bypass mode position:

the bypass damper is positioned to expose the one or more bypass air outlets; and

the air inlet damper is positioned to seal the room air; and wherein in the at least one air filtering position;

the bypass damper is positioned to seal the one or more bypass air outlets; and

the air inlet damper is positioned to expose the room air inlet.

12. The air purification system of claim 1, further comprising:

a first temperature sensor configured to automatically measure room air temperature and a second temperature sensor configured to measure supply duct air temperature.

13. The air purification system of claim 7, further comprising:

a temperature sensor configured to automatically measure room air temperature; and

wherein the bypass damper system is configured to be in the at least one air filtering mode position when a first temperature condition is met; and

wherein the bypass damper system is configured to be in the second air filtering mode position when a second temperature condition is met.

14. The air purification system of claim 1, further comprising:

a first ionizer configured to automatically emit charged particles into the air in a blend chamber located before the air filter and a second ionizer configured to emit charged particles into a supply duct chamber.

15. The air purification system of claim 1, further comprising:

a check damper configured to automatically close and stop contaminants on the filter from passing into the room air when the bypass damper system is in bypass mode position.

16. The air purification system of claim 1, further comprising:

one or more distribution panels configured to split air in two or more channels and provide distributed air across the filter when the bypass damper system is in the at least one air filtering mode position.

17. The air purification system of claim 16, further comprising:

a substantially V-shaped load balancer configured to provide roughly equal amounts air to each of the two or more channels.

18. The air purification system of claim 1, further comprising:

a first temperature sensor configured to automatically measure room air temperature;

a second temperature sensor configured to automatically measure supply duct temperature; and

an electronic controller configured to log room air temperatures from the first temperature sensor and log supply duct air temperatures from the second temperature sensor, the electronic controller further configured to determine and provide user alert messages based on the logged temperatures.

19. The air purification system of claim 18, where in the user alert messages include one or more of: an air filter changing recommendation, a temperature setting recommendation, a repair/service recommendation, a room door opening recommendation, and a refrigerant replacement/addition recommendation.

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20. The air purification system of claim 18, wherein the second temperature sensor is located in the supply duct air inlet.

21. The air purification system of claim 1, further comprising:

a first pressure sensor configured to automatically sense both supply duct pressure and ambient room air pressure;

an electronic controller configured to log supply duct pressure and ambient room air pressure from the first pressure sensor, the electronic controller further configured to determine and provide user alert messages based on the logged pressures.

22. The air purification system of claim 21, wherein the electronic controller logs both room door open and room door closed ambient room air pressures from the first pressure sensor.

23. The air purification system of claim 21, wherein the electronic controller logs supply duct air pressure for a diagnostic period of time at one or more of the following conditions: without a filter installed, with a new filter installed; and with a used filter installed.

24. The air purification system of claim 22, wherein the first pressure sensor is configured to automatically establish open door and closed door initial pressure readings during installation; and wherein the electronic controller is configured to utilize the initial pressure readings to calculate an open door room pressure threshold and a closed door room pressure threshold.

25. The air purification system of claim 24, where in the electronic controller is utilizes the initial pressure readings in determining the user alert messages.

26. A method of providing pressurization relief comprising:

providing an air purification system configured to fluidly receive air from a forced air system, to fluidly receive air from a room, and to expel filtered air into the room; measuring air pressures from the forced air system supply duct;

measuring room air pressures;

measuring room air temperatures;

measuring air temperatures from the force air system supply duct;

automatically releasing unfiltered air from the forced air system into the room when the air received from the forced air system exceeds a bypass pressure threshold; automatically expelling filtered air from both the room and the forced air system when the air received from the forced air system does not exceed a bypass pressure threshold and a first temperature condition is met; and automatically expelling filtered air from only the forced air system into the room when the air received from the forced air system does not exceed a bypass pressure threshold and a second temperature condition is met.

27. The method of claim 26, further comprising:

automatically releasing unfiltered air from the forced air system into the room when the system is off or malfunctions.

28. The method of claim 26, further comprising:

automatically activating a fan at high speed, when the air received from the supply duct inlet exceeds a minimum pressure threshold, lower than the bypass pressure threshold.

29. The method of claim 26, further comprising:

automatically resetting to expelling filtered air when the bypass pressure threshold is no longer exceeded.

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**30.** A method of providing usage alerts to a user of an air purification system comprising:

installing an air purification system configured to fluidly receive air from a forced air system, to fluidly receive air from a room, and to expel filtered air into the room;

measuring installation air pressures from the forced air system supply duct;

measuring installation room air pressures;

automatically determining a room profile including one or more room pressure thresholds utilizing the measured installation pressures; and

measuring post installation room air pressures;

measuring post installation air pressures from the forced air system supply duct; and

providing an alert to the user when a room pressure and/or supply duct pressure threshold is reached.

**31.** The method of claim **30**, wherein:

the installation pressure reading(s) include one or more of: a pressure with a new filter, a pressure without filter, and a pressure with a used filter; and

each respective type of installation pressure reading is utilized to calculate a respective room pressure and/or supply duct pressure threshold; and

each respective room pressure and/or supply duct pressure threshold triggers a corresponding alert to the user when the room pressure and/or supply duct pressure threshold is reached.

**32.** The method of claim **31**, wherein installation pressure reading(s) from the forced air system supply duct are measured for a diagnostic period of time.

**33.** The method of claim **31**, wherein installation pressure readings without a filter, are utilized to calculate a first supply duct pressure threshold corresponding to a first alert that

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informs the consumer of the utility cost impact of a type of filter being used in the forced air system.

**34.** The method of claim **31**, wherein installation pressure readings with a new filter, are utilized to calculate a second supply duct pressure threshold corresponding to a second alert that informs the user of the need to replace the forced air systems used air filter with a new one.

**35.** The method of claim **31**, wherein installation pressure readings with a used filter, are utilized to calculate a third supply duct pressure threshold corresponding to a third alert that informs the user of a potential malfunction of the forced air system.

**36.** The method of claim **30**, further comprising:

measuring and logging room air temperatures;

measuring and logging air temperatures from the forced air system supply duct; and

providing user alert messages based on the logged temperatures.

**37.** The method of claim **36**, wherein the user alert messages include one or more of: an air filter changing recommendation, a temperature setting recommendation, a repair/service recommendation, a room door opening recommendation, and a refrigerant replacement/addition recommendation.

**38.** The method of claim **30**, comprising measuring both room door open and room door closed installation room air pressures.

**39.** The method of claim **38**, wherein room open door and closed door installation pressures are utilized to determine an open door room pressure threshold and a closed door room pressure threshold.

\* \* \* \* \*

UNITED STATES PATENT AND TRADEMARK OFFICE  
**CERTIFICATE OF CORRECTION**

PATENT NO. : 9,254,459 B2  
APPLICATION NO. : 14/029650  
DATED : February 9, 2016  
INVENTOR(S) : Miller

Page 1 of 1

It is certified that error appears in the above-identified patent and that said Letters Patent is hereby corrected as shown below:

In the claims,

Claim 9, column 26, line 48, please delete “wherein the a” and insert --wherein the--;

Claim 11, column 26, line 65, please delete “wherein the a” and insert --wherein the--;

Claim 25, column 28, line 31, please delete “controller is utilizes” and insert --controller utilizes--;

Claim 26, column 28, line 42, please delete “the force air” and insert --the forced air--.

Signed and Sealed this  
Third Day of May, 2016



Michelle K. Lee  
*Director of the United States Patent and Trademark Office*